AN INVESTIGATION of the RISKS of EXPLOSION in the OPERATION of FLAME_PROOF

MINING APPARATUS.

by

G. HIBBERD, A.R.T.C.

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Kirkgreen Miniskisk 27/12/29. The Very Revend Professor S. Helligan, D.D., D.C.L., bleck of mate, The University, Islasgow, DearSir, adverting to your letter in which I was instructed to amend my thesis. An investigation of the Risks of Caplosion in the Operation of Flame. Poool Mining upparatus and submit it on as before 31th Dec. 1929, I have honestly endearoured to satisfy the requirements, and now submit my work for adjudication imended as shows: 1) an introduction is given explaining the reasons is undertaking the research, and showing the relationship it bears to the work of investigation who may have had a similar and in view. See pages 1-14. See Lages 1-IV. (2) The tabulated statements of result have been arranged an indicated an 'page V. a sepoint of a paper published in the 4th Number ! (3)

of The Tournal of The Royal Technical College, and a mathematical investigation of The Drop in Temperature Produced by the Passage of Hot Gases through apertures are given in appendise i, Jop. 84 - 88. a bibliography is given in appendix ii, pp. 89-90. (\mathcal{A})

Tousting that my work, thus amended, meets with approval I am, scar Sis, Yours Respectfully,

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AN INVESTIGATION of the RISKS of EXPLOSION in the OPERATION of FLAME_PROOF MINING APPARATUS.

By G. Hibberd, A.R.T.C.

INTRODUCTION: In May, 1924, the author was awarded the Walter Duncan Research Scholarship by the Governors of the Royal Technical College, Glasgow, and with the approval of the Board of Studies of the College decided to investigate certain problems connected with the safe working of electri: :cal machinery in mines.

> At that time the term "flame-proof" was applied somewhat loosely to certain types of machinery intended for use underground and many machines considered flame-proof were later shown to be unsafe. This was chiefly due to the lack of suitable references for the guidance of designers and manufacturers of this type of machinery.

> The problem of protecting apparatus was attacked by many eminent investigators, and in searching for a solution, the character of flame and flame movement, the various methods of initiating by electrical means an explosion of gaseous mixtures and methods of preventing excessive rise of pressure in a gaseous explosion in a partially closed vessel had to be thoroughly investigated before the real problem of adequate protection could be approached.

> In this preliminary work the experimenters used spherical bronze vessels, e.g., in February 1924, in a paper read before the Institution of Mining Engineers, Professor Douglas Hay and Mr. Ira C.F. Statham gave an account of experiments, carried out in such vessels, and also mentioned that they had tested a large number of switch boxes etc., intended for use in mines. A short time later the Safety in Mines Research Board Paper No. 5 appeared under the joint authorship of I.C.F. Statham and R.V. Wheeler. This paper

also gave results of investigations in which spherical bronze vessels were used, and in addition put forward certain recommendations which they considered necessary so that flame: -proof conditions might obtain.

These papers were welcomed and it was recognised that a great step had been made towards the solution of the problem of safe working of electrical machinery in mines. It was felt, however, that the results obtained were only truly applicable to spherical vessels, and some difficulty arose with regard to the application of the recommendations to casings such as were already in use in mines. These casings had been evolved from long experience of the requirements having special regard to economy of space and material, and the ability to withstand the rough usage associated with mining operations. Naturally they differed greatly in material and shape from the spherical vessels used by Hay, Statham and Wheeler.

Many manufacturers had independently evolved machinery which they claimed to be flame-proof, but even after the publication of the above-mentioned papers some doubt existed among them with regard to the margin of safety of their products. This was clearly seen when Sheffield University announced that they were prepared to receive and test Flame-Proof Apparatus and issue certificates if the apparatus satisfactorily passed the tests to which they were submitted. A large number of manufacturers availed them: :selves of the opportunity and now possess certificates for the apparatus which passed the required tests. In the opin: : ion of many, testing of such apparatus after application of recommendations should not be necessary. Testing might be resorted to as a precautionary measure but recommendations issued should be such that after their application the possibility of failure should not exist.

The author, therefore, decided, after careful

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perusal of available literature (a bibliography is given in an appendix) and after consideration of the results and recommendations given by investigators to confine himself to a research in which the vessels used would be typical examples of modern mining switch-gear casings, and to endeavour to present results in such a form that designers and others interested would have no difficulty in deciding the dimensions of flanges, gaps, etc., for casings which had to be flame-proof.

Flanged casings were obtained and experiments performed as here described. The results obtained have been presented in the form of graphs from which a suitable design for the flange of any casing and a suitable depth for the flame-proof gap can readily be found.

Abnormal conditions such as the "blowing" of a fuse under large overload conditions, the presence of coal dust in addition to gas, the effect of increased temperature of casings due to working conditions or electrical defects have all been considered. In addition the possibility of passage of flame at the bearings of protected motors has been fully dealt with.

So far as the author is aware these problems have not been discussed in any paper published in this country, and he is confident that the present work will be a useful addition to the list of publications dealing with Flame-Proof Apparatus.

- SUMMARY: I. The work was undertaken because of the necessity of presenting results in a manner that could easily be understood and applied with confidence to actual mining switch-gear casings, etc.
 - 2. This could only be done by experimenting with casings such as were found most suitable (with regard to design and material) for use in mines.

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3. The problems dealt with are entirely practical and practical solutions are offered.

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4. The work bears no distinct relationship to other works which have the same object in view - viz: absolute safety in operation under all conditions likely to be encoun: :tered - but forms what should be a useful addition to the list of works on the subject of "Flame-Proof Apparatus", since it deals with problems about which very little was hitherto known.

In Appendix No. i a reprint of a paper by the author published in the 4th Number of The Journal of The Royal Technical College - is presented together with a mathematical analysis of the drop in temperature produced by the passage of hot gases through apertures.

In Appendix No. ii a list is given of the more important works perused by the author. The writer is much indebted to their authors and freely acknowledges the assistance he received by studying their papers. In this connection it is desired to pay special tribute to the work of Professor W.M. Thornton, D.Sc., D. Eng., whose many excellent papers were a constant source of inspiration.

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AN INVESTIGATION OF THE RISKS OF EXPLOSIONS IN THE OPERATION OF FLAME_PROOF MINING APPARATUS.

Electricity by reason of its adaptability is now used extensively in coal mines, but its introduction has been accompanied by many dangers, chief among which are the possibility of ignition of coal dust and inflammable mixtures of methane and air (firedamp). A parallel potential source of ignition may be said to exist in the flame safety lamp, but the protection afforded by gauzes is such that complete confidence may be placed in modern types of lamps, and the design is such that omission of parts during assembly is inexcusable.

Unfortunately absolute reliance, in electrical appara: :tus for use underground, is not justified at present, from a safety point of view. Many protective devices have been introduced, (quite a number displaying the main features of the safety lamp) to guard against the possibility of ignition of gas and dust, but with all devices, great care in inspection maintenance and assembly of parts must be exercised if the apparatus to which they are fitted is to remain safe.

Many vague ideas exist regarding the exact nature and function of these devices, consequently, protected apparatus may become unsafe through deterioration, neglect or lack of knowledge of proper methods of maintenance of these contrivanc: :es.

Few of the original problems of protection of apparatus remain to be solved, and progress in this direction is such that the day is not far distant when it will be possible to install Apparatus, safe in every respect, and be assured that it will retain its safety features as long as the apparatus is in commission.

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The present Research was conducted under the direction of the Professors of Mining and Electrical Engineering in the Royal Technical College, Glasgow, and is chiefly concerned with such methode as might be applied to underground electrical Apparatus in order to afford adequate protection from explosive mixtures of gas and air.

Demonstrations were given annually to the Senior Engineering Students of the College.

THEORY OF PROTECTION: Occurrence of Methane:-

The most common explosive gas mixture found in coal mines consists of methane and air, and is generally referred to as "firedamp." Methane is given off naturally as the coal is worked. The gas was probably entrapped during the early stages in the formation of coal, but dissociation of coal substance by great pressure and high temperatures acting over long periods of time, may partly account for its presence. If the gas were allowed to accumulate highly explosive mixtures would in all probability be formed so that to keep mixtures within eafe limits dilution with air is necessary.

The "Coal Mines Act" requires that "a sufficient quantity of air shall be constantly circulated to dilute and render harmless all noxicus and inflammable gases" to such an extent that the "orkings, shall be in a fit state for working and passing therein."

Under normal conditions every endeavour is made to main: thain the required standards of ventilation, but abnormal occurrences may allow of accumulation of gas where it is least desired e.g. in the vicinity of electrical apparatus. If the system of ventilation is interrupted, say, by a fall of material in an airway, or a stoppage of the fan, gas will accumulate in dangerous proportions in "gassy" sections of the mine. The workings of the mine may also be flooded by the sudden liberation of large quantities of gas

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from an "outburst." In addition to the above the presence of "blowers" of gas necessitate special attention to ventilation if the formation of explosive mixtures of gas and air is to be prevented.

PRESENCE OF GAS IN APPARATUS .-

Electrical apparatus is subjected to variation of temperature during and subsequent to working periods. Expansion and contraction of the metal casings and gaseous contents will accompany each rise and fall in temperature. It follows therefore that in the process of cooling after working periods, gas will be "drawn" into the box or casing, through the joints, to maintain equilibrium of pressure with the outside. If the surrounding atmosphere contains methane, a highly inclammable mixture may thus be introduced into the casing and thereby create the required conditions for an explosion. Openings in the casing may permit of natural diffusion of gases, so that independent of temperature variations gas may accumulate in electrical apparatus. If the above conditions exist all that is necessary to initiate an internal explosion is a spark or electrical arc, and in event of flames being communicated to the outside of the apparatus, an explosion having far reaching and serious effects may be the final result. Gas tight apparatus or hermatically scaled casings would seem to be the remedy, since accumulation of gases inside casings would be prevented, but difficulty of inspection, renewal and repair, together with rough usage, and deterioration of sealing meterial in the humid atmospheres of mines, render the application of this measure impracticable. In the absence of sealing methods we cannot prevent the accumulation of gas in casings, so that the danger of ignition is ever present. The real danger lies in the transmission of high temperature gases and flames to the outside, and all protective methods aim at the prevention of this.

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IGNITION OF FIREDAMP: -

The mixtures of C.H.L (Methane) and air which on burn: ing evolve the greatest amount of heat contain approximately 9.5 per cent of methane, and are highly explosive. In these mixtures the methane and oxygen are present in approximately the proportionenecessary for complete chemical combination. If the percentage of methane be reduced below 9.5 per cent less heat will be generated by the burning mixtures since obviously the amount of heat will depend on the amount of inflammable gas present. In mixtures containing over 9.5 per cent of methane, there is insuffic: tient oxygen present to effect complete combustion. If the per: contage of methane be continuously reduced, a point is reached at which only sufficient heat is generated by the burning portion to raise an adjacent portion to the ignition temperature. With further reduction flame propagation does not take place. A similar state of affairs is reached by increasing the percentage of methane above 9.5 per cent. The points at which self propagation of flame ceases are termed the "limits of inflammability" and are 5.3 per cent and 14.8 per cent respectively. It is evident that any variation in the condition of experiment which alters the amount of heat transferred from the burning portion to an adjacent portion will alter the limits of inflammability. Factors which affect the limits are: - material and dimensions of containing vessel, initial temperature and pressure of the mixture, and state of mixture i.e. whether turbulent or quiescent.

Wheeler has shown that the most explosive mixtures are not the most readily ignitable mixtures, and also that slow butning mixtures will transmit flame through small passages when more Tapidly burning mixtures would not.

Firedamp has a peculiar property by virtue of which a source of ignition must be in contact with an inflammable mixture for a definite period of time before ignition takes place, e.g. if the temperature of the source be 6_{70}° C. 10 seconds elapse between the introduction of the source and ignition of mixture; if the

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source is at a temperature of 1000°C ignition occurs in less than one second. The time is not occupied in bringing the mixture to the temperature of the source (this is probably effected in a small fraction of a second) but may be due to the rate at which chemical combination takes place. The rate of combination is quite measurable at 300°C and increases rapidly with temperature. This combination results in liberation of heat which raises the temper: sature of the surrounding mixture. Thus the rate of combination increases rapidly until finally the temperature is such that combination takes place throughout the mixture almost instantaneous: :ly. It follows therefore that the higher the temperature of the source the shorter will be the period necessary to raise the temperature of the gas to that at which instantaneous combination is effected throughout the mixture. This is important from a mining point of view, since if flame or hot gases from an explosion in a casing, can be cooled considerably before reaching the outside, the chances of external ignition will be slight. The flames from an explosion are of short duration, and if considerably cooled, would not be in contact with the external mixture for the period of time necessary to initiate explosion at the temperature of the "dooled" gases.

POSSIBLE SOURCES OF IGNITION: -

It has been shown that any sustained flame, however small, will cause ignition of firedamp mixtures. The arc produced at a switch or controller when a circuit is opened, is of short duration but the temperature is very high and is quite capable of causing ignition of gas mixtures. Gas surrounding a fuse will be ignited if the fuse is "blown." Sparking at commutators of D.C. motors or slip rings of A. C. motors must be regarded as a potential source of ignition. Broken colls in stators and rotors of induction motors must also be regarded as dangerous, when gas is present.

TRANSMISSION OF FLAKES FROM APPARATUS TO THE OUTSIDE: -

Under the conditions described above, if the motor or

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switch gear is not protected external explosion with its accompany: ing destructive effects will inevitably result. If the apparatus is adequately protected the possibility of external ignition is very great. If the casings are protected but constructed of material not sufficiently strong to withstand the force of the explosion, the conditions are equivalent to no protection. It is therefore apparent that in addition to preventing passage of flame a casing must be able to withstand the pressure developed by ang explosion.

GAS EXPLOSIONS IN CLOSED VESSELS: -

When an explosive mixture of methane and air is present in a metal casing and a source of ignition is applied(say an electric spark), flame spreads rapidly from the point of ignition throughout the mixture at a rate depending on the amount of methane present. Actual measurement of the temperature of explosion have recorded 1900° C - 2100° C as the highest temperature attained. Theoretically, the temperature should be considerably higher, but the proximity of the cold walls of the containing vessel and probable dissociation of methane at high temperatures, each absorbing heat, probably together account for difference between the theoretical and recorded temperatures. Since excessive temperature development is prevented, pressure development is restrained in a like manner.

The maximum pressure recorded for methane-air explosions in completely closed vessels, is 102 lbs per sq. inch. If the joints of the vessel be separated by a small amount the pressure is reduced considerably, since part of the mixture is expelled unburnt through the space at the jointe, when ignition takes place. Hot gases produced by an explosion will also escape if a gap is left at the joints of the casing.

METHODS OF PROTECTION ADOPTED: -

From the foregoing it will be evident that the provision of a large cooling surface is essential, and in methods of protection adopted, this aspect has been kept in view. In a few

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- 6 -

types of devices, cooling of flame has apparently been the principal object aimed at, and the possibility of obtaining considerable reduction of pressure while effectively cooling the flame has not received the attention it merits. Methods of protection of apparatus have so far proceeded mainly along four lines. One of the earliest methods made use of gauze and perforated plates as protective devices. These devices are founded on sound physical principles and involve the provision of large cooling surfaces condisting of wire gauze. The gauzes were separated by distance pieces, but numerous layers were necessary before the desired effect was achieved. In early applications gauzes were used for cooling purposes only; the attainment of considerable reduction of preasure seems to have been lost eight of, with the result that one of the necessary requirements of adequate protection was not obtained. In addition coal dust was free to enter the apparatus and also lodged on the gauzes and in the interstices, interrupting ventilation. The gauzes would be readily corroded, so that frequent inspection and renewal would probably be necessary, Recent research has shown that if properly applied, and entry of coal dust prevented by layers of tin or lead foil, satisfactory results may be obtained and considerable lowering of pressure be produced in event of explosion.

Labyrinth and plate protection was also tried. The device consisted of large numbers of thin metal plates so arranged and spaced that escaping gases and flames had to traverse long tortucass passages before reaching the outside atmosphere. In passing through the labyrinth, effective cooling was obtained but such was the resistance offered to the escaping gases that only slight reduction of pressure resulted. This appliance requires accurate machining and fitting, and is easily damaged. Coal dust will lodge between the plates and reduce the efficiency of the device. This device has not been extensively applied for the above reasons.

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Other contrivances adopted are known as relief values. They consist of values, normally held closed by springs, set to open a specified distance immediately the pressure rises above a certain amount. In passing through the values the hot gases are brought into contact with large cooling surfaces. These devices are also most effective flame coolers, but fail to cause appreciable reduction of pressure.

Special devices have special dissivantages, but dis: advantages common to all of them may be enumerated as follows.

- The addition of devices to any apparatus means additional care in inspection and maintenance of apparatus, which of course increases cost of upkeep.
 - Machining of parts and application to casings tends to increasing price of product.
- 3. Special devices require expert handling, and ignorance or carelessness in handling may render the device useless and the apparatus unsafe.
- 4. Accumulation of dirt, grease, rust, etc., may hinder the proper functioning of the device.
- 5. Concentration of escaping gases at small area is not desirable.
- 6. Doubtful whether flame-cooling and pressure reduction can each be carried to safe limits in the same device.

FLANGE PROTECTION: -

2.

This is perhaps the simplest and most effective method of cooling gases and lowering pressure of explosion.

It consists of providing broad flanges at all joints in casings, the flanges being so proportioned that the required degree of cooling is obtained, and by having a gap of predetermined depth between the flanges, excessive pressure development is prevented. The great advantages of flanges are simplicity low cost and perman: ience. There is nothing that can go out of order, and maintenance and renewal costs are nil. The U.S. Bureau of Mines advocate metal to metal contact of not less than one inch at all joints. This of course means high pressure in event of an explosion, and necessitates very strong and rugged casings. No advantage is taken of the property called "lag on ignition." All that is required is cooling of flame to 700°C or thereby, and as already pointed out the lag on ignition at this temperature is such that external ignition cannot take place since the temperature is not maintained for a sufficient length of time.

Since flange protection in our opinion offers the greatest possibilities for adequate protection of electrical apparatus for use underground, it was chosen for this particular investigation, and all casings used in the experiments were fitted with broad flanges.

FLAME PROOF ENCLOSURE: -

The "British Engineering Standards Association" give the following definition of a flame-proof enclosure:- "A flame proof enclosure is one that will withstand without injury any explosion that may occur within it under conditions of the rating of the apparatus (and recognized overloads, if any associated therewith) and will prevent the transmission of flames such as will ignite any inflammable mixture which may be present in the surround: :ing atmosphere."

THE FLAME-PROOF TEST: -

The following test is applied to casings to determine whether they satisfy the requirements of the above definition:-The apparatus to be tested is placed in an "Explosion Chamber" and an explosive gas mixture is introduced into it. The Explosion Chamber is now closed and filled with an explosive gas mixture, after which the mixture in the casing is ignited and the result noted. If the external mixture (i.e. the mixture in the chamber surrounding the apparatus) is ignited, flame has passed from the apparatus, and

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under the conditions of the test, it is unsafe. If no external ignition occurs, the mixture is ignited by some means, to prove its explosibility. The tests are repeated until a consecutive number of similar results are obtained, and according to results the apparatus is declared asfe or unsafe under the conditions of the test.

THE EXPLOSION CHAMBER: -

The "Explosion Chamber" used in the investigation consisted of a wooden box, fitted with terminals and a gas tap, and of the following dimensions: - Length 2 ft. 10 ins. Width 2 ft. 7 inches, Height 2ft. 6 ins. To prevent damage in event of an external ignition, safety values or windows were fitted into the ends of the box. These windows were 15 ins square and during the filling stage were blocked with paper to prevent leakage. When external ignition occurred the paper windows were burst, allowing of dissipation of energy without damage to the box. If external ignition did not occur a light on the end of a stick was thrust through the paper to ignite the mixture and prove its explosibility.

METHOD OF OBTAINING EXPLOSIVE MIXTURES: -

The explosive mixtures were obtained by mixing coal-gas and air in known proportions. Objection to this might be made on the ground that coal gas is not found in mines, and that firedamp, the explosive mixture generally found in mines, should have been used instead. It may be pointed out however that the surrounding gas mixture is really only an indicator the purpose of which is to indicate when flame has passed from the internal explosion. Evident: :ly then a mixture of coal-gas and air will serve the purpose admirably and such a mixture has the additional advantage that it more sensitive to ignition than methane and air (firedamp). Consid: :ered in this light results obtained, using coal gas and air, will not vary greatly from those obtained using methane and air and any difference will be in favour of coal gas since a first margin of safety all be obtained.

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METHOD OF IGNITING THE EXPLOSIVE MIXTURES: -

The gas mixtures in the casing under test were ignited electrically by "blowing" a fuse, power being taken from a 250 wolt. D.C. supply with a variable resistance in circuit. The fuse wire was No. 24 S.W.G. Tin and from 12 to 2 ins.in length.

APPARATUS TESTED IN THE EXPERIMENTS: -

Each piece of apparatus tested in the experiments was fitted with fixed flanges at the joints of cover and casing. The casings were typical examples of modern mining switch-gear casings. These casings were readily adapted for experimental purposes, by tapping the sides or ends to receive a gas tap and a lead-in plug for the conductors. This adaptation did not in any way interfere with the protective features of the boxes. The use of typical switch gear casings is a step which commends itself, in that any weakness or defect in the casings could be readily detected and suggestions made for general improvement.

EXPERIMENTAL: JUNCTION BOX WITH GROOVED FLANGES: -

The first set of experiments was carried out on a junction box, detached from a gate-end circuit breaker. This box had a grooved flange 0.5 ins broad, into which fitted a cotton gasket, so that when the cover was bolted into position a very tight joint was secured. The ends of the box were blocked with pieces of wood, bored to receive terminals and a gas pipe. Leads were taken from the terminals inside the explosion chamber to those on the junction box, and short leads from the latter were bridged with a piece of fuse wire, inside the junction box. These connections having been made, the cover was bolted into position, an explosive gas mixture was introduced, and the gas pipe plugged to prevent leakage. The "windows" of the saplosion chamber were now blocked with paper and the junction box surfounded with an explosive atmosphere. By closing the switch the fuse was "blown", igniting the gas in the junction box, the effect produced being noted. A large number of tests as above ware made with the gasket

in position and then with it removed. Typical results are shown in the following table.

TABLE 1.

Tests Made on Junction Box with Grooved Flanges 0.5 ins.

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Boi	K - 1		Explosic Chamber.		t of Flange.		Ignition. nal.External	Remarks.
.0:	20.		3.400.	25	0.5	Yes.	No.	Tape in position in Groove.
	M		ei	5	R.	2 P	n	п
	q		li	R	h	n	17	It
		11.	1990 - A	1 - 1 = 0				
		-	17. ja 19.	2018 Jan 1999			la	Signs of Charring.
	Π		ti .	ŧ	π	Ħ	Yēa.	Tape removed from Groove.
	8		n	B	71	Ħ	u	n
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CONCLUSIONS: -

It was found that with the gasket in position flame did not pass between the flanges, but after a few ignitions distinct signs of charring were seen, showing that if the experiments were continued the gasket might possibly be burned out, and thereby provide a passage for flame to the outside mixture. With the gasket removed external ignition took place in every imperiment, indicating that a flange 0.5" broad is not in itself in adequate protective device. In tape or gasket is merbanically weak and would soon rot



in the humid atmosphere of mines, and for this reason alone could scarcely be recommended.

If for any reason the gasket had to be removed great care would have to be exercised in replacing it correctly in position in the groove, to prevent risk of providing passage for flame. Above ground with abundant light this would require ordinary care only, but underground where lighting arrangements are often far from perfect it would be a more difficult matter.

It would be better to avoid the dangers which might arise, by removing the gasket altogether and increasing the breadth of flange to one inch or more if necessary. If this measure were adopted, we would have a piece of apparatus made flame proof at the beginning and always retaining its properties, unlike/a device depending for its flame proof properties on a material which readily deteriorates and hence gradually loses its properties.

EXPERIVENTAL: - PRELIMINARY: -

1

GATE END CIRCUIT - BREAKER WITH ROUGH MACHINED FLANGES: -

The next piece of apparatus, was a gate-end box fitted with circuit breakers set to trip at 17.3 amperes. Part of the gear was removed to adapt the box for experimental purposes. As shown in the photograph the cover is hinged and held in position by means of eight Sinch. diameter bolts. The flanges were 13 inches broad and rough machined. The junction box used in the above experiments was re-attached to the gate-end box and tests similar to those already described were carried out.

After one or two tests in which external ignition did not occur, it was thought that the combined junction box and gate end box was quite safe under the conditions of the test. To confirm this, one more experiment under exactly the same conditions as the others was made. A violent external explosion resulted when the switch was closed, and on removing the junction box cover the cotton gasket was found to be completely based through in parts, and badly charred in others. The flame proof properties of the

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combined apparatus were thus entirely destroyed. The junction box was removed and a piece of wood, bored to receive a gas pipe was bolted into position in its place.

A number of experiments were now carried out with the flanges of the cover and box in contact, the only gaps being those formed by the rough machining, the largest of which did not exceed .01 inches. Under these conditions it was not found possible to initiate an external explosion. By placing pieces of metal between the flanges it was possible to obtain gaps of any desired depth all round the perimeter of the cover. Experiments were thus carried out at various depths of gap until external ignition was produced by flame passing between the flanges from an internal ignition. The following table shows the average results of a large number of experiments.

TABLE 2.

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Tests on Gate End Circuit Breaker. Box. Flanges 13 ins.

Broad. Volume 1.55 Cub. ft.

Gate-	all all and	Igniting Current in	Breadth of	of	Igniti Internal. H		1 Romarka
0.350		28	1.75ina.	0.025	ins. Yes.	No.	Ext. Mixture Exp.
0.40 0	3.600			4	н. н. н	- 58.3	n
-	3.686		and an	H	п	-34	n
0.419	4.043	15 15 Jul	an logan	0.0361	na. "	. • •	
0.420	4.039	1 1		N	5	н	
					2 9 2		
0,490	4.400	e e artis (a. s	Ħ	0.0551	13: 	¥es.	Very Violent Expl. Chamber Shattered

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In the last experiment of this series the Explosion Chamber was shattered. Apparently the "safety valve" paper windows did not operate rapidly enough to prevent damage to the box. A gap of 0.055 inches all round the Gate End Box would provide a large release area, and at that particular depth of gap the flanges could not have a great cooling effect on the flame from the internal explosion. When the internal ignition occurred a large mass of intensely hot flame would be projected into the external mixture from all round the box. The pressure wave preceding the flame would create considerable mechanical disturbance in the external mixture and followed immediately by a large body of flame, ignition would take place under conditions approaching detonation. Under these abnormal conditions the pressure rise would be so rapid that maximum pressure would be attained before relief was obtained by rupturing of the paper safety valves. The energy of the explosion would therefore be expended on the sides of the explosion chamber instead of being dissipated through the "safety valves." The result was of course destruction of the explosion chamber since it was not designed to withstand such pressures.

COOLING EFFECT OF FLANGES ON FLAME: - -

To obtain some information on the flame that causes ignition of the external mixtures, the following experiments were performed with the gate-end box in the "open", i.e. surrounded by air. An explosive mixture was admitted to "box" in the usual manner and ignited, the distance the flame extended beyond the flanges being noted approximately. These tests were performed with the same depths of gap as were used when the box was in the explosion chamber. With the exception of the first series, i.e. with flanges in contact with each other, flame was observed in every case, often blue in colour and extending only a short distance beyond the flanges. This was chiefly noticeable with gaps up to 0.025 inches in depth. In these cases geoling had been carried

store "to concern the second of surgers

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out to a successful degree, and in some of the experiments in which weak mixtures were used the flame seemed to emerge at a relatively slow rate, and never exceeded one inch in length. At gaps greater than 0.025 inch the flame was brownish-red in colour and extended 3-4 inches. At gaps at which external ignition was known to occur bright red flames streaked with white, extended more than six inches beyond the flanges, when ignition took place. The flame was visible all round the casing. <u>IGNITION OF GAS WIXTURES AT "WAKING" and "BREAKING" CONTACT in the</u> (GATE.)

END BOX: -

The gear removed from the box was now replaced and a number of experiments carried out to determine whether an explosion of gas could be initiated at "making" or breaking contact. A piece of strong twine was attached to the handle of the gate-end switch (which was in the OFF position), an explosive mixture was admitted to the box, then after closing the other switches, contact was made in the gate-end box by pulling the cord. The circuitbreakers were set to trip at 17.3 amperes, but as there were only 9 ohms. resistance in the circuit the short circuit current would be 27.8 amperes. It follows therefore that no sconer was contact made than it was broken by the circuit breakers operating. The flash thus produced ignited the gas mixtures in the casing. Extra resistance was introduced and the current lowered to 15 amperes. With this reduced ourrent ignition occurred in each experiment at "break", but on one occasion ignition took place at "make". This was probably due to dirtycontacts, because when they were cleaned no further ignitions occurred at "make."

CONCLUSIONS:-

The combined gate-end box and junction box may be represented as two separate chambers with a short inter-communicat: ting passage. It has been shown that the operation of the overload releases causes ignition of a gas mixture, and in event of such an cocurrence in the combined apparatus the wixture in the smaller

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Er 2 - SECTION SHOWING LOOSE LOOSE FLANGES IN FORMING A GAR POSITION













chamber would be compressed before ignition so that higher pressures would be developed. This probably accounts for the sudden rupturung and burning of the octton gasket. With a piece of wood covering the opening over which the junction box is normally attached, the gate-end box is flame proof with gaps up to 0.038 ins. in depth between the flanges all round the perimeter. With the studs in position and the cover secured, there is little danger of an external explosion if the opening referred to above is well covered. In passing, it might be well to mention that the stud hole at the hings communicates with the inside of the casings (the others do not). It therefore follows that if this stud is omitted the cooling properties of the flanges are of no avail because we have a direct passage from the inside of the box to the outside atmosphere. This could be prevented by bottoming the stud hole, i.e. by covering the bottom of hole with metal as shown in the fig. 5. If this alteration were made the value of the box from a flame-proof point of view would be considerably increased. The failure of the combined apparatus by the burning of the gasket is a good illustration of how adequate protection of a piece of apparatus (such as by broad flanges of gate-end box) may be rendered useless by an inadequately protected piece of apparatus (junction box with narrow flange and gasket) which is connected to it. The remedy is protection to the same degree of safety for all parts of same apparatus, always remembering that adequate protection implies sufficient strength to withstand the explosion and ability to cool the gases from the explosion. CRITICAL LENGTH OF OPENING: -

In this series of experiments an effort was made to determine the maximum length of opening for any particular depth of gap between flanges of given breadth, which prevents passage of flame capable of igniting an explosive mixture of gases. The apparatus used consisted of two switch boxes of different volumes and fitted with flanges of different breadths. The volume of the smaller box was approximately 0.120 oub. ft. that of the larger

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2.352 cub. ft., while the former was fitted with flanges 1 inch broad and the latter with flanges 12 inch broad. The flanges were smooth machined.

The smaller box was used in the first part of the investigation. Loose sheet metal flanges (0.010 inch thick) of shape shown in fig. 3. were prepared and placed between the flanges of the cover and the box, so that when the nuts were tightened, the only passage from the inside of the box to the outside atmosphere was through a small rectangular opening. The length of the gap could be fixed and the depth altered by adding or removing loose flanges. Fig 2. shows this clearly.

The initial depth of gap was 0.1 inch, and it was found that before flame passed at a temperature sufficiently high to ignite an explosive mixture the gap had to be grinch long.

Experiments such as above were carried out with gaps of 0.08 inch, 0.06 inch, 0.042 inch etc., until 0.030 inch was reached when it was found that this depth of gap could be left all round the box and external ignition could not be produced by the flame from an internal explosion.

Tests similar to those described above were carried out on the larger box special loose flanges being prepared for the purpose.

The gate-end box already described was now adapted so that experiments similar to those above might be made on it. By so doing a better comparison was obtained and the effect of breadth of flange and volume better understood.

The results from this series of experiments are shown numbered and tabulated at the end of the thesis, while in table III selected results are given to demonstrate the manner in which the critical length varies with depth of gap and breadth of flange. <u>CONCLUSIONS AND DEDUCTIONS:-</u>

This series of experiments was most interesting especially when nearing what might be termed the "critical length of opening" i.e. the length of opening which, at any given depth

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CRITICAL LENGTHS OF OPENING FOR FIXED DEPTHS OF GAP.

TABLE II.

ENDINE	Switter	AMPONEC	FLANGE	Levern	DEPTH	NTEPN	N TEPN E TER AL		
3.408	_	29	1		001-	Yes	No	EXTERNAL MIXTURE EXPLOSIVE	E EXPLOSIVE
3-0 6	550+	12	•	.875	080.	Yes	٥٧	Do	Do
2.25	350.			1.750	-058	YES	Na	Ďα	De
3.270	020.	:		6.50	270-	Yes	No	Do	Do
24	.030		+	9.00	055	Yes	No	Do	Do
5-110	0:0.	÷	•	2 4.000	550.	Yes	Yes		
2.000	0201	.r	Ŧ	t	020.	Yes	No	Excession Misture	EXPLOSIVE.
010-E	607.	15	1-375	1.000	001-	Yes	No	Do	Do
2.700	0.05:	4	1	2.875	02 0:	Yes	No	Do	D.o
2-850	312	=	Ŧ	9.500	-052	455	No	Do	Do
2776	+315	-		70.000	070.	Yes	Yes		
2700	.302	:	÷	:		Yes	No	EXTERNAL MATURE EXPLOSIVE.	LAPLESIVE.
24.00	515.				020	Yes	No	Do	Do
2.000	072-	•	1.750	005.7	960.	Yes	No	Do	Do
2.600	058.	÷	+	6.250	080.	Yes	No.	Φ	Do
2.400	280		z	12+125	1056	Yes	Na	Da	Do
2.000	075-	4	+	15:435	10.45	Yes	No	Do	D _o
2-600	354			(Feergeres)	. 058	Yes	No	D_{0}	Do

LENGTH OF GAP ? THE LENGTH OF OPENING [NEASURED ALONG THE FLANGE] LEFT FOR ESCAPE OF HAT GASES

DEPTH OF GAP ? THE DISTANCE BETWEEN THE FLANGE OF BUN AND FLANGE OF COVER AT THE OPENING LEFT FOR ESCAPEOF

HET GASES

of gap, just prevente passage of igniting flame. A small altera: :tion of length of opening at this point decides whether external ignition will occur or not. We pass from a stage where there is quite definitely internal ignition only, to a stage where there is definite external ignition occurring apparently simultaneously with internal ignition, so short is the interval between them. Between these stages we have what might be termed a "retarded ignifion" stage in which the "lag on ignition" effect is often observed. In this stage two distinct reports are heard, first the internal explosion followed by the external explosion. The area of opening has been slightly increased but cooling is still being carried to a considerable degree but not sufficient to lower the temperature of the gases below the ignition temperature of the surrounding mixture. In other words the "lag" at the degree of cooling is not of sufficient duration to prevent external ignition.

It is inconceivable that, at the short lengths of opening cooling is produced solely by contact with the flanges since only a small area is exposed to the flames and the time of contact is of short duration. The rapid expansion of the hot gases from the explosion through a small opening, will result in considerable cooling, and at these small lengths of opening must be regarded as the predominant factor in preventing ignition of the external mixture. At the critical lengths cooling is carried just sufficiently far to prevent ignition of the outside mixture. consequently any slight alteration of length will affect the degree of cooling. An increase in length increases the area through which expansion takes place. It follows therefore that expansion to atmospheric pressure will take place at a relatively less rate since the velocity of the issuing gases will be reduced as the area of opening is increased. The slower rate of expansion will maintain the temperature of the gases for a relatively longer period, so that they will issue at temperatures at which the "lag"

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is negligible and ignition of the external mixture will result. In the "retarded ignition" stage the temperature is maintained just sufficiently long to allow of ignition of the external mixture at the particular temperature, and the distinct pause between the two explosions indicates the effect of "lag on ignition."

As the gap is reduced in depth and increased in length, a greater area of flange is exposed, so that at the smaller depths of gap cooling by contact with flanges will predominate. When we reach a depth of gap which can be safely left all round the casing, cooling of the gases from the internal explosion will be obiefly accomplished by the flanges and cooling by expansion will be negligible.

CURVES: -

From the tabulated results three curves were plotted with abscissae "Critical Length of Opening", and ordinates "Depth of Gap". (Fig. 6.) The curves are hyperbols? with the XY ordinate as asymptotes. From log. curves equations were obtained for each curve and by extrapolation the curves were extended beyond the points obtained in the experiments.

If we regard the perimeter of each box as a critical length, and, using the equations which satisfy the curves, calculate the depths of gaps which should prevent passage of flame at the particular perimeter, we find in each case that the calculated depth is much less than that found by experiment. The reason for this difference may be explained as follows. When a gap is left all round a box we have what might be termed an unrestricted passage to the outside. Under these conditions, when ignition occurs, the rapid expansion of gas at the source of ignition will expel unburned gas near the opening. This expulsion of unburned gas will occur at all lengths of opening, and when the length of opening is the perimeter of the box the loss must be considerable, since gas will escape in all directions. The heat generated depends on the amount of combustible gas present, so it naturally follows that under the conditions described the heat given off from the

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Loc D.

burning mixture will be very much less than the potential heat of the original mixture, and the amount of heat to be dissipated will be considerably less than when the casing is completely closed or only partly open.

The large surface provided by the flanges will readily cool the escaping gases and it will be possible to increase the depth of gap up to a point beyond the calculated depth, and still prevent passage of flame capable of causing ignition of the external mixture. Increased depth of gap will increase the area of opening and tend to increase the loss of gas at ignition, but a "depth" will be reached when only the gas in contact with the flanges will be greatly cooled, and the temperature of the gas in the centre of the passage will not be appreciably lowered. In other words the flanges will be so far separated that self propagation of flame in the gas mixture between them will be possible. It follows therefore that when this point is reached external ignition will result. The depth found by experiment therefore probably represents the maximum 'depth' to prevent self-propagation of flame between the flanges, and for the reasons stated above will be greater than the calculated value, when the perimeter is regarded as the critical length.

The maximum 'perimeter gap' (i.e. maximum gap that can be safely left all round the casing) for each box is indicated by a straight line parallel to the X-axis and intersecting the curve. It does not follow that these are the safe gaps for these flanges for all volumes and perimeters; e.g. taking the smallest vessel - volume .123 cub. ft. breadth of flange 1 inch perimeter 29 inches, safe perimeter gap .030 inch. - we cannot say, that this depth of gap is safe for any casing fitted with 1 inch flanges. Casings of larger volume than .123 cub. ft. will invariably have openings with perimeters, greater than 29 inches, since they will house larger gear. Greater volume means greater potential heat for same strength of gas mixture so more heat will have to be dissipated, and if breadth of flange is to remain at 1 inch the safe perimeter gap may have to be less than .030 inch. The curves

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show that increase of perimeter (where perimeter is regarded as a critical length) necessitate a decrease in depth of gap for same breadth of flange and volume, consequently if both volume and perimeter increase while the breadth of flange remains the same, a further reduction in depth of gap will be necessary.

The alternative to reduction of depth of gap is an increase in breadth of flanges, and is indeed the more satisfactory method of retaining flame-proofness. The increase in size and weight of casings will, in general, necessitate an increase in breadth of flanges at joints for reasons other than preventing passage of flame, e.g. increase in size of holes for studs, or bolts, to retain covers in position etc. However for casings to be used in mines the flame-proof factor is the most important. It would be interesting and useful if the relationship between volume and breadth of flange could be shown by a curve or expressed by an equation.

In fig. 7 a series of curves is shown, abscissae are "Critical Lengths of Opening," and ordinates are "Breadth of Flange." Readings are taken for same depths of gap from the ourves in fig. 6. The curves indicate that breadth of flange has a great effect in preventing passage of flame. For example, if we take a gap of 0.1 inch depth, we find that with flanges 1 inch broad the critical length is 0.375 inch, while with flanges 1.75 inch broad the critical length for the same depth of gap is 4.0 inches an increase of 3.625 inches in critical length for an increase in breadth of flange of 0.75 inches. In this set of curves the effect of volume of casing has not been considered.

Fig. 8 shows another series of curves in which ordinates and abscissae are "Volume of Casing" and Critical Length respective: :ly. Readings for equal depths of gap are again taken from fig. 6.

Although no account is taken of breadth of flange the influence of this factor is clearly seen, since the box with the broadest flange has the greatest length of gap for a given depth of gap.



VOLUME OF CASING IN CUBIC FEET

We cannot draw any important conclusions from these two sets of curves, since in each, no account has been taken of factors having an important influence on prevention of passage of flame.

If we consider the volume left for passage of hot gases etc., (i.e. the product of length of gap, depth of gap, and breadth of flange) we have here a figure which includes the main determining factors in preventing passage of flame. If this were plotted against "volume of casing" a satisfactory relationship would not be shown, since the curves obtained would be somewhat similar to those shown in fig. 8. This of course is due to the fact that the largest casing used in the experiments was not provided with the broadest flange consequently its "volume of passage" was not the greatest for a given length of gap.

If, however we express the "volume of passage" as a ratio of "volume of casing", and plot this against volume of casing we obtain curves which will show in a sarisfactory manner the relationship existing between these two factors for any given length of gap.

As the "volume of passage" is very small compared with the "volume of the casing", the figure representing the ratio will have a very small value.

We can overcome this difficulty by expressing the "volume of passage" in cubic inches and the "volume of casing" in cubic feet the result being the, "volume of passage" in cubic inches, required for each cubic foot of "volume of casing," and any particular length of gap.

Calculations on this basis have been made and a table compiled (table IV), from which the curves shown in fig. 9 were plotted, dipwing relationship for lengths of gap between 1 inch and 100 inches.

In the calculations "lengths of gap" are regarded as "critical lengths", it follows therefore, that the "Volume of passage" given for any particular" length of gap", is that which just prevents passage of flame capable of igniting an external

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[•] VOLUME OF CASING IN CUBIC FEET



FIG. 10. - BREADTH OF FLANGE REQUIRED FOR GIVEN VOLUME OF PASSAGE AT LENGTHS OF GAP BETWEEN I INCH AND 100 INCHES

BRERDTH OF FLANGE IN INCHES

TABLE I

CALCULATED VALUES OF 'V' AND R FOR VALUES OF 'L BETWEEN I" AND 100

		and the second		And in case of some line in the local sectors.		
LENGTH OF	DEPTH OF	AREA OF	BREADTH	VOLUME OF		R - VOLVME OF PASSAGE
GAP [L]	GAR (D)	GAP (A)	Of FLANGE (F.)	PASSAGE	OF CASING (V2)	VOLUME D. CALING
IN INCHES	IN INCHES	IN So Inches	IN INCHES	(A×E)ins ^a	IN CUBIC FEET	= V/ V2
,	076	.071	1000	.074	-/23	· 57Y
	103	.103	1.375	·/43	2.352	.060
	133	+135	1.750	. 221	1.550	. 14 2
	251	.153	1.000	·153	•/23	1.243
3	075	.225	1.375	· 309	2.352.	./27
	-097	.291	1.750	.509	1.550	328
6	.043	.258	1.000	.258	-123	2.097
	.061	.366	1-375	.503	2.352	.009
	.079	.474	1.750	.829	1.550	-534
10	.037	.37	1.000	.370	.123	3.008
	.053	-53	1.375	.730	2.352	• 374
	.068	-68	1.750	1.190	1.550	·767
00	030	.60	1.000	.60	-123	4.878
20	043	.86	1-375	1.18	2.352	-501
	056	1.12	1.750	1-96	1.550	1.264
70	027	181	1.000	.810	.123	6.585
30	038	1.14	/ 375	1.560	2.352	.662
-	050	1.50	1.7 5 0	2.600	1.550	1.677
40	024	.96	1.000	.960	123	7.804
1	035	1.40	1·375	1.920	2.352	· 802
	045	1.80	1.750	3.100	1.550	2.000
50	023	1.15	1.000	1.150	·123	9.352
	033	1.65	1.375	2.270	2 352	.465
	.043	2.15	1.750	3 760	1.550	2.425
60	.021	1.26	1.000	1.260	123	10-225
	.031	1.86	1 375	2.550	2.352	1+084
	·040	2.40	1.750	2.200	1.550	2.709
70	. 020	1.40	1-000	1.400	.123	11-263
	.030	2.10	/·375	2.900	2.352	1.232
- 14	.039	2.73	1 750	4 770	1-550	3+077
80	.019	1.52	1-000	1.520	.123	12.357
+	- 028	2.24	1-375	3.0B0	2352	1.309
	.037	2.96	1.750	5.180	1.550	3.277
90	.019	1.71	1.000	1.71	·123	13.902
-	. 028	2.52	1.375	3-46	2 352	1+ 471
	.036	3.24	1.750	5.67	1-550	3.6.58
100	1018	1.80	1.000	1.800	. 123	14 634
	.027	2.70	1 375	3.700	2.352	1: 5 75
	035	3.50	1.750	6.120	1.550	3-947

inflammable gaseous mixture. Consequently if we fix the perimeter of a casing of given volume (and regard the perimeter as a "critical length "), reference to the curves will give the "volume of passage" in cubic inches per cubic foot of "volume of casing," required at the length of gap, equivalent to the perimeter, if flame-proof conditions are to hold. Having obtained this figure, the total "volume of passage" for the casing under consideration is readily found.

In fig. 10 a family of curves is shown, abscissae being total "Volume of passage," and ordinates "Breadth of flange." The volume of passage is shown for lengths of opening between 1 inch and 100 inches.

It naturally follows that if we obtain the "volume of passage" required for any particular perimeter and volume from fig. 9. reference to fig. 10 will indicate the breadth of flange necessary under the circumstances, and knowing the perimeter, breadth of flange, and volume of passage, the permissible depth of "perimeter gap" (i.e. gap left between flanges all round opening) may be easily found.

Figs. 9 and 10, should prove of great use to designers of flame-proof casings, when some doubt exists regarding the proportioning of breadth of flange, and depth of gap, for any casing, the volume and perimeter of which have been fixed, by the electrical gear it is intended to house.

Three worked examples are given to illustrate the use of the curvest-

Casings of the following dimensions are required to house certain electrical gears-

(1)	Volume	0.2cubic	feet,	opening	at a	cover, 9	1nches	by	6 intee
(2)	- 3-5 - 8	0.6	1 14			* ,12	the state		8 •
(3)	•	1.4	н.,			• ,18		י 1	.2 🏿

Find for each the breadth of flange necessary and the maximum permissible depth of perimeter gap in order that the

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casing may be flame proof.

Now R = $\frac{V_1}{V_2}$. V_1 = RV2 Let Volume of passage in cub.ins. = V1 * ft. = V_2 * casing And $V_1 = FxLxD$. $D = V_2$ ExF. 🖬 R Vo and L. are given. Let R and F are found from figs. Let breadth of flange in inches. = F 9 and 10. V1 and D are calculated. perimeter of casing in inches= L R D gap (1) $V_2 = 0.2 \text{ ft.}^3$, L = 2(9 + 6) = 30 ins.From fig. 9., R, for L = 30 ins, and $V_2 = 0.2$ ft.³ is given as 5 ins³/ft.³ $V_1 = RV_2 = 5 in.^3/ft.^3 \times 0.2 ft.^3 = 1.0 ins.^3$ From fig. 10, F, for L = 30 ins and V_1 = 1.0 in.³ is given as 1.1 inch. $D = \frac{V_1}{L = F}$ = $\frac{1.0 \text{ ins.}^3}{30 \text{ tr}}$.030 ine. 30 in. x 1.1 ins. .'. breadth of flange required is 1.1 inch and permissible perimeter gap is 0.030 inch. (max.) (2) V₂ = 0.6 ft.³, L = 2(12 + 8) = 40 ins. From fig. 9., R, for L = 40 in. and $V_2 = 0.5 \text{ ft.}^2$ is given as 4.05 in.3/ft.3 $V_1 = RV_2 = 4.05 \text{ in.}^3/\text{ft.}^3 \pm 0.6 \text{ ft.}^3 = 2.43 \text{ in}$ ig. 10, F, for L = 40 ins. and $V_1 = 2.43 \text{ in.}^3$ is given as 1.55 ins. From fig. 10, F, for L = ins. 3 $= \frac{V_1}{LF} = \frac{2.43 \text{ in.}}{40 \text{ ins. x } 1.55 \text{ ins.}}$ = .039 ins. ... breadth of flange required is 1.55 and permissible perimeter gap is 0.039 ins. (max.) $V_2 = 1.4$ ft.³ L = 2(18+12) = 60 ins. (3) From fig. 9, R, for L = 60 ins and $V_2 = 1.4$ ft.³ is given as 3.0 in.3/ft.3 $V_1 = RV_2 = 3.0 \text{ in.}^3/\text{ft.}^3 \times 1.4 \text{ ft.}^3 = 4.2 \text{ in.}^3$ From fig. 10, F, for L = 60 ins. and V_1 =4.2 in.³ is given as 1.75 ins. $\frac{v_1}{LF} = 4.2 \text{ in.}^3$ D = - .040 ins. 60 \$ 1.75 breadth of flange required is 1.75 inches and permissible perimeter gap is 0.040 ins.(max.)

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CORL GAS IN EXPLOSIVE MITTURE IN CUBIC FEET.

IGNITING CURRENT AND PAS AGE OF FLAME: -

This part of the "research" was devoted to investigating the effect of igniting currents of various strengths, on mixtures of gas near the limits of inflammability.

The small switch box (volume 0.125 cub .ft. flanges 1 incl broad) was used in the experiments. A gap 0.053 inch in depth was chosen, the critical length being 1.625 inches.

The procedure was exactly similar to that already described; an explosive mixture was introduced into the small box, the latter was now placed in the Explosion Chamber and surrounded with an explosive atmosphere, after which the switches were closed, thereby igniting the internal mixture. The results were noted.

By keeping the current constant throughout a series of experiments, in which the gas mixtures were varied from the lower to the upper explosive limits, points were obtained above and below the most explosive mixture, at which external ignition could not be produced, although the mixturesignited was within the explosive range.

On repeating the above experiments, using a higher value of igniting current other points were obtained at which flame capable of causing external ignition no longer passed.

The points so obtained might be conveniently referred to as the "flame passing limits", and the range of mixtures between the points as the "flame passing range."

The currents used in the experiments varied from 3 amperes to 78 amperes, and it was found that the "flame passing range" increased until igniting current had a value of 30 amperes. The "range" did not increase above this value of igniting current, and was found to be nearly coincident with the "explosive range".

Two sets of points, representing the "lower" and "upper" limits "of the "flame passing range", were thus obtained and are shown plotted in fig. II.

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A few of the results are given in the following table.

TABLE V.

The Effect of Strength of Igniting Current on Passage of Flame from Explosions of Gas and Air Mixtures.

	n oubio		Bdth.	G	ap.	Ignitic	on.	
S.B.		Cur. Ampa.	of Fl.	Depth.	Length.	Internal.	External	Remarks
.027	2.700	3.5	1=	.058	1.625 +	Yes.	No.	
.028	11	3.5	R		R	N	Yes.	
.027	Ħ	14.5				a	Yes	
.025		14.5	8	ri -	н		No	
.024	8	33.	h	•		н	No	Lower
.025	A	33					Хеа	Limit.
.025		60			•		Yes	
.024	æ	78		ų			No	
.031	2.700	16	1*	.058	1.625 +	Yes	Yea	
.032	н	16	E.				No	
.032	•	23	•				Yes	11-section 1
.033	•	23		e			No	Upper Limit.
.0325	5 •	40				B	Yes	
.034		40		a	R		No	
.033	•	60	•				No	
.034		78		n			No	

CONCLUSIONS & DEDUCTIONS: -

The opening between the flanges was the minimum length of opening for passage of flame at the depth of gap used, consequently increase or reduction of the amount of coal gas in the mixture, by reducing the potential heat of the mixture would in itself be sufficient to prevent passage of flame.

The existence of the "flame passing range" is not surprising. As already pointed out any alteration of the conditions of experiment when working with mixtures near the explosive limits,

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Berthelot in his experiment observed that the explosibil: :ity of mixtures near the limits was increased, when ignition was accompanied by shook which produced mechanical disturbance of the mixture. It follows therefore that as the igniting current is increased the disturbance set up when the fuse is blown will increase. so the rate of propagation of flame will increase and time rate of development of pressure will be relatively greater. Weak mixtures will therefore burn with greater rapidity as the igniting current is increased, and conditions may be such that flame capable of causing ignition will be projected through the opening between the flames. Points will be reached above and below the most explosive mixture where the mechanical disturb: :ance is no longer capable of producing such marked effects, and these points will be very near, if not coincident with, the explosive limits of the mixtures used.

In the experiments the range was a maximum when the igniting current reached a value of 40 amperes, and increase of Value of current beyond this did not widen the range. In Fig. 11 the results obtained are shown in the form of a graph.

Since the phenomenon is observable only at limit mixtures under abnormal igniting conditions and is apparent only when the length is increased beyond the oritical length for the particular gap, it follows that if the gap is such that flams from the most explosive mixtures is effectively cooled in passing through the gap no fear need be entertained regarding the possibility of external ignition by fuses blowing under considerable overload.

A few experiments similar to those described above were carried out with gaps of 0.078 inch and 0.040 inch, and the results, in general, confirmed those already obtained.

COAL DUST:-

The effect produced by addition of coal dust to the gas mixtures.

The question of ignition of coal dust in addition to

gas mixtures is one that must be considered. Where designed openings are left in casings, entry of fine coal dust will inavitably occur, if the apparatus is used in a coal mine. In all electrical apparatus, its presence is undesirable, since faulty electrical contact etc. will be produced, if the quantity of dust is excessive. The dust will also lodge in the opening and might prevent, or hinder, the effective operation of protective measures.

In order to investigate the effect of the presence of coal dust in the casing and between the flonges, in addition to explosive gas mixtures, and to determine whether the conclusions arrived at using gas mixtures alone, would have to be modified to embrace the effect of added dust, a large number of experiments were carried out.

A sample of coal was obtained and ground to pass through a 200 mesh sieve.

Proximate analysis performed in duplicate gave the following average results.

Volatile Matter	(Volatile Hydrocarbons24.2%
Coke	(Moisture
	(Ash
	Fixed Carbon. 62.5%

QUANTITY OF DUST ADDED: -

It was decided to add dust in the proportion of one ounce per oublo foot of air space, a quantity much in excess of that necessary for propagation of flame in a dusty atmosphere. The reason was, of course, to make the test as severe as possible.

The box used in the experiments was the small box already described, having volume of 0.12 cub. ft., and flanges one inch broad. The gap chosen was 0.058 inch deep and 1.6254 inches long. This gap produces external ignition with gas mixtures.

In the initial experiments dust was placed between the flanges at the gap. It was found to have no appreciable effect, because when ignition of dust and gas took place inside, the casing, the dust between the flanges was blown out and was not

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ignited. The placing of dust between the flanges was therefore abandoned on this account.

The most severe conditions possible under the circumstances, were imposed on the box but only in a few cases out of a large number of experiments was it possible to obtain external ignition. In two of the cases external ignition was effected through the bursting of the box by the internal ignition, direct communication with the outside mixture being thus afforded. The bolts securing the cover of the casing were cocasionally sheared also.

When the cover was removed after an experiment, dense yellowish fumes, the product of partial distillation of the coal, were found. It would acome therefore that this evolution of distilla: ition products etc., had some effect in preventing passage of flame. If however we consider the bursting of the two boxes, and the number of bolts that were sheared, we might be inclined to suggest that dust added to the gas mixture increases the pressure attained by the explosion.

We might visualise what takes place in the box when coal dust and coal gas present in explosive proportion are ignited. Here we have two combustible substances and a certain amount of When the fuse "blows", ignition of the gas mixture cocurs and air. a dust cloud is raised. Part of this dust must be consumed directly, part will be volatilised, while a certain amount will be expelled, probably unburned, from the casing. The products of this partial distillation cannot be consumed since the oxygen has been used up in the explosion of gas and burning of dust, so that the box will be filled with spent gases and volatile matter not a very suitable medium for flame propagation. The mixture that issues from the box will therefore donsist of coal dust, distillation products and gases from the explosion. Condensation and cooling of the contents of the box immediately follows, and part of the surrounding mixture will rush in through the opening. The mixture in the box may now be inflammable but the temperature

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TABLE VI.

PRESSURES DEVELOPED BY EXPLOSIVE GAS MIETURES UNDER VARIOUS CONDITIONS

CUBIC FRET	GNITING	GAP		PRES	Sures	AVERAGE	
OF COAL GAS	IN AMPERES	LENGTH			••• •	₽/-3 	Pressures Ibs/a
· 015	18	.375	.10 "	5	5	5	5
.020		+		20	22	22	21.5
.025		+		39	39	38	39
-028		•		41.5	42	41	41.5
-030	*	-	•	40	44	43	42
032		н	•	41	10	40	40
·055		•		32	32	33	32.3.
-040	ŧr			23	20	22	22
.042		-	· • .	$ \mu $	12	11	11.3
.015	4	.175	080	5	5	5	5
.020				18	2 3	22	21
.025	4			34	34	33	34 -
.027	•	•	- 4	34	35	3 8	36
·030		. •	н. 1	36	37	40	38
·035		+		32	33	33	32.6
- 040	•	*		24	18	25	22.3
.045	н	-	. *	5	5	5	5
.020	-	1.625	.060	21	20	20	20
025			•	33	32	31	32
· 0 30		144	4	36	35	35	35.3
-035				29	32	30	30.3
·040		*	- fai	21	19	20	20
• 0 2 0	••	6.0	·040	18	18	18	18
·025			· + .	26	30	28	28
030	14		1	34	35	32	33
· 035	- 44	*	-4	24	32	27	28
-04 0	-4	•		18	18	18	18
·020	. 4	9.0	.036	3	3	4	3 3.
025		- 11	1	14	15	18	16
· 03 0	- 11	•	*	21	21	18	20
· 03 5				12	14	15	14 -
.040	. 14	4		3	3	4	3-3.
.030	- 44	29" PERMETERI	030	6	6	5	6 -
-030		1625	-010 "	39			39
.0.20			030	37			35
.0 30	4		040	34			34
030	*	-	.058	33			33
.020			-080	29			29.

has fallen below the ignition temperature, so further ignition does not occur.

It is therefore evident that the dust cloud raised has a damping effect on the flame of the gas explosion, and the partial distillation will cause a further fall in temperature, so that gases issuing from an explosion will have a temperature considerably lower than those issuing from an explosion of a gas mixture only.

The quantity of coal dust equivalent to one ounce per cubic foot of air space, amounted to 3.5 gms. This amount of dust was practically all consumed by ten internal ignitions. Of the ten ignitions, only the last one, in general produced external ignition, being practically a gas ignition.

CONCLUSIONS: -

To sum up, we might say that the results obtained indicate that the addition of coal dust does not increase the danger of passage of flame, but rather retards it, while the pressure developed by the explosion is increased.

MEASUREMENT OF PRESSURE DEVELOPED BY EXPLOSIONS: -

An important part of this investigation lies in the measurement of pressures developed by explosions of gas and air mixtures. So far, the only point considered has been the prevention of passage of flame capable of igniting the external mixture. It is apparent however that results are incomplete without some knowledge of the pressures developed at the various 'critical lengths.'

We have already mentioned that adequate pressure release and prevention of passage of flame are the sime of all protective measures. From a "flame-proof" point of view these two factors directly oppose each other, for, while a large escape area lowers pressure it increases danger of passage of flame, and a small escape area reduces possibility of passage of flame but tends to development of high pressures. Both factors must therefore be considered when we intend to leave designed gaps between flanges, since obviously, the most suitable gap will be that

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COAL GAS	COAL DUST [2001] ESH]	IGNITING Current	GA		PRESSURES Recorde o In Ibs/0"			Average Pressures
CUBIC FRET	GRAMS.	AMPERES	LENGTH	DEPTH		1		lbs/=
·030	NIL	21	6	-04 0	35	31	35	33
·030	+	26			32	33		
• 03 0		35			34	33	32	33
·030		42		+	33	32.5	32	3 2 .5
·030		50		**	34	32	33	33
.030	-14	60		** K	33	34	32	33
· 015	3.5	18	1.625	·058	9	8	8	8.3
· 020	3.5	- q					19	20
.025	3.5	er.		•				31
.038	3.5		4		36	37	37	36.6
-029	3.5		- 14	4	37	38	39	38
· 0.30	3.5	4		•	39	39	39	39
• 03/	3.5	4	- 4		38	38	38	38
1035	3.5	4			32	33	34	33
-040	3.5			4	21	21	21	21
· 04 5	3.5				8	9	9	8.6
.015	NIL	"	7		7	7	У	7
· 020	4,	. 4	- 11		20	20	19	19.6
· 025	~	*	•		30	32	33	31.6
.028		+1			34	34	33	33.6
.029	4				34		35	34.6
.030	ı		**	.,	36	35	35	35.3
.03/				4	36	34	35	35
·035	- 44	-	4		29	32	30	30.3
· 040	•		· · ·	4	21	19	20	20
.045	•	- 4e			7	7	7	7.

TABLE <u>VII</u> Pressures Devaloped By Gas MixtOres Containing Coal Dust.







FIG 16 - REPRODUCTIONS OF TIME PRESSURE CURVES READ FROM RONT TO LETY]







FILM - LENGTH AND DEPTH OF GAP AND IGNITING CURRENT ALL KEPT

CONSTANT

which gives the lowest possible pressure, consistent with preservation of "flame proof" properties.

Now, as the critical length increases, the "escape area" (i.e. product of "critical length and "depth of gap) increases so that we may expect lowering of the pressure developed by the explosion of the internal mixture. It has already been shown that the flange area exposed, increases, and that cooling by contact with the flanges becomes predominant, as the "critical length" increases.

Apparently then, when cooling is brought about by contact with large metallic surfaces, we may expect greatly reduced pressures, whereas when the contact area between the flanges is small, higher pressures may be expected.

It has been pointed out that when the "escape area" is small, the flange area exposed is small, and that cooling is effect: ied mainly by expansion.

Under these circumspances, we must expect relatively bigh pressures.

In the measurement of the pressures three factors were considered viz, gap left between the flanges, gas quantity present, and value of igniting current. Each factor can be varied at will, and in the following experiments each has been varied in turn while the other two remained constant.

Thus, three sets of results were obtained.

- (a) Pressures developed with gas quantity and gap constant, and igniting current varied.
- (b) Pressures developed with gap and igniting current constant, and gas quantity varied.
- (c) Pressures developed with igniting ourrent and gas quantity constant, and gap varied.

The apparatus used was similar to the small switch box already referred to, and pressures were measured by means of a steam-engine indicator (see fig. 17.) No attempt was made to obtain time-pressure records in all the experiments, but in special

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DEPTH OF GAP IN INCHES FIG20, MAXIMUM PRESSURES DENELOPED AT CRITICAL LENGTHS experiments a few were obtained by realeasing the 'drum' of the indicator, at the moment of ignition. Typical indicator cards are reproduced in figs. 15 and 16.

The results of these experiments are best shown by means of graphs and in addition to the tabulated results curves are shown in figs, 12, 13 and 14, in which the effects of the various factors on pressure development can be readily seen.

When igniting ourrent is the variable factor the graph of pressure is a straight line paralel to the abscissa, indicating that the value of the igniting ourrent has not any effect on the ultimate maximum pressure of the explosion. This, of course, is what we might expect. The only effect we might look for, would be, that the pressure rise might be more rapid with a large current than with a small igniting ourrent.

With gap and current constant and gas quantity varied the effect on pressure is naturally, very marked. We traverse the entire explosive range of gas mixtures in this series, and observe how the pressure rises rapidly from the lower limit of explosibility to the most explosive mixture, and then, as the gas quantity is increased beyond this point the pressure falls rapidly until the upper limit is reached at which pressure is very low.

By keeping the gap length, current and gas quantity constant, and varying the depth of gap from 0.10 inch to 0.01 inch, a set of pressure records were obtained from which a curve was plotted. In this graph we observe particularly the effect of decreasing the depth of gap, or in this case continually reducing the "escape area" between the flanges. As the "escape area" is reduced we gradually approach the totally enclosed state in which no provision is made for escape of gases or pressure release, and bence pressure developed is a maximum. It follows therefore that as this state is approached the pressure must increase.

PRESSURES DEVELOPED AT CRITICAL LENGTHS: -

The pressures developed by different gas mixtures at critical lengths were also measured and curves plotted from the

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result (see fig. 19.) The ourves show that the pressure falls by small amounts until the gap is 0.04 inch. At depths of gap less than 0.04 inch the pressure fall is very rapid until we can leave a gap all round the perimeter. The pressure recorded under the latter conditions is only about 6 lbs per square inch.

We have referred to cooling by contact, and expansion, and described the conditions under which each predominates. There must, however, be a certain point at which each is equally effective, and above and below this point some characteristic must become apparent to indicate the factor which is chiefly responsible in effecting cooling of the gases from the explosion.

We have shown that pressure is a minimum when cooling by contact predominates, consequently when a point is reached at which pressures begin to fall rapidly it must be evident that the expansion effect no longer controls cooling.

The position of this point is readily detected in fig. 20 in which ordinates are "maximum pressures at oritical lengthe" and abscisses are depths of gap.

The point at which change occurs is 0.04 inch depth of gap. We may reasonably assume then, that cooling by contact begins to assert itself from .04 inch gap downwards and that at gaps over 0.04 inch, expansion is the chief cause of cooling, while the effect of each is apparently equal at this particular depth of gap.

PRESSURES DEVELOPED WHEN COAL DUST IS ADDED TO THE GAS MIXTURES: -

Coal dust, in the proportion used in previous experiments, was added to the box and a set of pressure records obtained. The results are shown plotted in fig. 14, and indicate that the pressure is higher than when gas mixtures alone are used. Under favourable conditions very much higher pressures than those indicated may be recorded.

GENERAL CONCLUSIONS: -

It has been shown that strength of igniting current does not affect the ultimate pressure attained by an explosion in

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a partially closed vessel.. We have also seen that the addition of coal dust increases the pressure developed. The most important results however are those showing the pressures developed at 'oritical lengths'. These results help materially in deciding which method of protection to adopt. It is clearly evident that cooling by contact with large cooling surfaces is preferable to cooling by expansion, or any method in which resistance is offered to the escaping gases, as, for example, in passing through a special device. It is of importance to note that large cooling surfaces, from which the 'escape area' is small, do not completely satisfy the requirements, because pressure developed varies inverse; ily as the 'escape area'. In employing flanges with the proper perimeter gap between them, we provide a large cooling surface and a large 'escape area', and, consequently combine effective cooling with adequate pressure release. In this respect flange protection is superior to all protective devices, because, as we have already pointed out, it is doubtful whether the two factor can be satisfactorily combined in the same appliance.

WEBBED COVERS: -

While carrying out a series of experiments in which a stronger type of box was used it was observed that critical lengths were slightly greater than in the other casings of same dimensions. The breadth of flange was the same in each case, and the volumes were also approximately equal, so that we might have expected similar critical lengths. The material of which each casing was made was cast iron, therefore possibility of different rates of conduction of heat cannot be entertained.

In the stronger type of box there were several internal webs. This, while strengthening the box increased the internal surface area by approximately 30 sq. inches. It follows therefore that a larger cooling surface was presented to the hot gases of the explosion so that the temperature reached would be less than in the box in which there were no webs. The gases issuing at the opening would be at a lower temperature than under the old

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FIC21 - SMALL SWITCH-BOX WITH HEATING COIL AND THERMO-COUPLE IN POSITION AS USED IN HEATED CASING EXPERIMENTS



TEMPERATURE OF CASING

INITIAL TEMPERATURE OF CASING 55"F

conditions, therefore an increase in the 'escape area" would be necessary before flame passed to ignite the external mixture.

The critical lengths would therefore be greater in the stronger box.

The increase in length was slight but very marked. Further investigation is necessary before definite conclusions can be arrived at, but the results are certainly indicative of what might be done in the direction of cooling gases before they pass between flanges or through protective devices.

Internal, and external webs, are already applied to casings of certain machinery used in mines, the purpose of the application being to prevent excessive rise in temperature under working conditions. The idea could be readily extended to switch-gear casings etc., to increase the cooling surface present: :ed to hot gases in event of an explosion occurring, and thereby considerably assist in maintaining the flame-proof features of the apparatus.

HEATED CASINGS: -

Up to the present the experiments described have all been performed with the casings at the atmospheric temperatures prevailing at the time of experiment. Strictly speaking therefore, the results obtained, and the conclusions arrived at, apply only to ordinary atmospheric conditions.

The rise in temperature of electrical apparatus depends on the working conditions, and may be such that temperature of 100°C and over may be reached.

Electrical defects in the apparatus, will of course give rise to abnormally high temperatures at certain parts of casings, and in the following discussion are considered apart from temperature rise due to working conditions.

The problem that confronts us, is, to determine whether apparatus, flame-proof at atmospheric temperatures, still retains its properties at higher temperatures. If the properties are not

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FIG23: THERMO-COUPLE CALIBRATION CURVES

retained, we must consider what modifications are necessary to counteract the effects of the increase in temperature.

Now, ample cooling surface is essential in all protective measures, and the lower the temperature of the surface, the more efficiently will it function.

Each degree rise in temperature, will therefore reduce the efficiency of the cooling surface, and a point will ultimately be reached at which the cooling power will be negligible, so far as the cooling of hot gases is concerned.

As the temperature of the walls of the casing increase, their cooling effect on the gases, at ignition will not be so marked, and higher temperatures may be reached by the gases, if the vessel is totally closed.

The behaviour of the gaseous contents of the casing, under an increase in temperature, must also be taken into account.

Let us consider a totally closed vessel, into which a gaseous mixture at atmospheric temperature and pressure, has been introduced. Ageording to Charles Law if the temperature of the casing be raised, the pressure of the gases will rise above atmospheric pressure, while the volume remains practically constant. If the contents of the vessel were explosive, higher pressures would be developed on ignition, under the latter conditions.

Now, if the vessel were only partly closed, and communi: cation with the outside was effected, say through a small opening between flanges, totally different results would be obtained.

Free expansion is possible in this case and pressure remains constant. It follows therefore that with each degree rise in temperature a certain volume of gas , will leave the vessel. The volume of gas remaining in the vessel will be the same (approximately) at all temperatures, but the nett weight of gas will be reduced as the temperature rises. We see therefore that if the original mixture was combustible, it follows that as



FIG. 24 - HEATING AND GOOLING CURVES. NOTE EFFECT OF INITIAL TEMPERATURE OF CASING ON RATE OF HEATING temperature rises, the potential heat of the mixture remaining, decreases. (See fig. 22.)

It is thus evident that while the efficiency of the cooling surface is reduced with rise in temperature, in the event of an explosion occurring in the vessel, less heat has to be dissipated, so that it may still be capable of preserving the flame-proof properties so essential to safeworking.

In the case of cooling by expansion through a small opening or gap, we must consider the state of the mixture rather than the state of the vessel containing it.

We have seen that the potential heat of the mixture decreases as the temperature of the casing rises, therefore pressure developed by the explosion will be less, consequently the rate of expansion will be reduced.

The decrease in rate of expansion will of course be reflected in the range of cooling effected, and were it not for the fact that the potential heat of the mixture is less than normal, we might expect the temperatures of the expelled gasea to be higher than when the initial temperature was that of the surrounding atmosphere.

In the actual experimental work the greatest difficulty was to devise a suitable method of heating the casing to the desired degree and maintaining it at that temperature.

Obviously the most suitable means was electrical, and a ni-ohrome coil similar to those fitted in electrical fires was used. Temperature records were taken with a Copper-Constantan thermo-couple, calibration curves are shown in fig. 23.

The small box already referred to was adapted for the experiments, being suitably bored to receive the heating coil and the thermo-mouple. The arrangement is shown in Fig. 21 and Was found to be the most suitable under the circumstances.

In the initial experiments the usual explosive mixture was admitted to the box at atmospheric temperature, then heating operations were commenced, and continued until the required degree

METER	Galvano-	TEMPERATURE	GALINA		1112	Terpheresee +C		Der	Here e	HEATING CURRENT ANDERES
DEFLECTIONS	ET.	*C,	DS*4TIME	Rener	28 J.	and the second second	Destroy		COLUMN 1	the state of the local division of the local
			15	40	0.0	100	27	35	95	NIL
10	950	20	14	-	_	90	28	355	-	
10	-	26	14.5		_	85	29	32	-	۲
20	-	38	13	1*		74	30	31	E.	- Pr.
30	-	37	10.5	N		67	51	30.5		76
40	1	44	8.5		-	57	52	30	-	161
50	- 199	52	7	-		51.5	35	29		-
60		57	6			47.5	34	24	-	
70		64	5			41	35	29	-	
80	1 .	72	3.5			34	36	245	•	4.
80	-	70	2.5	1.0	-	26	37	28		
ALC: NO.		and the second second	2			and the owner of the local division of the l	38	o designation of the local division of the l		
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40	-	44	9+	"		89.5	41	26	6	
30		35	9-			45	42	26	•	- 16
10		24	7.5	. 4		78	43	25	-	- <u>4</u>
6		20	6			67	44	25		- 14
5	+	19	5			57.5	15	24.5	6	- F
4	4	15.5	45			51	HITIAL BAAVE P	TEMR (IF CA	SING IN S Has 15°C
o	2750	14+5	40			A8	1	30	93	2.0
4		22.5	3	Ι,		405	2	45	- 10-	
9.5		34	2.5		_	34	3	54		
		TAXABLE INCOME.	1.5		-	The other Designation of the local division of the local divisiono	and the second s	1000	-	-
15	,	155				26	4	62		
21		55	10		_	22	5	66	*	-
26	4	44-5	0	1 14		17	6	70		· •
32	- ÷	75	(-	1		llen	7	73	•	
36.5		43	Time IN			HEATING CURRENT	8	75	- U.	•
41	1 *	91	Mannes	Der:	Res.	AMPRES	9	77		
.46		99	1		93		10	79	-	-46
46		100	2	24	-		10.5	80	4	CAT OFF
44	2	96	.3	38	74		11	77	N	M.L
30	1 - 1	73.5	- 4	46	4	н	12	70	4	÷.
\$6.5	-	67.5	5	52	41		13	64		-1
24	-	63.5	6	37	Ar.	4	14	a second		. 4.
22		59	7	61						
		54-5	8	65	b.		15	65		
19			_	-	- 10		16	53		
16		49	9	67	*		17	50	•	
14.5	-4	46	10	70	11		18	48	•	••
11	- * -	40	"	75		1. N	19	47		
9	-10	35	12	74			20	45.5	4	<u> </u>
6		30	13	77		- *	21	44		1.0
5.5		27.5	14	79	-		22	43	1	10.00
_2		210	14-5	80	41	CUTOPE	ALTINA 1	PERIMA	P C4	SING IN WAS 320
1	+	19.5	15	75	т. С	Nih				
0		150	16	65						
6	4000	44	17	58	-					
\$		53-5	15	52	-44	-				
9		60			-					
	-		14	47	-					
11		67	20	<i>k</i> 4	-	-				
13	1	75	21	41		· ·				
14		81	22	139	-	•				
15	· · ·	\$5	23	37.5		-1				
16	340	90	24	36	1					
17		94	25	35						
		96.5		1.1.1	1					

. 26 34 +

96.5 ч - 5

of temperature was reached. During the heating stage gas was admitted to the explosion chamber till the correct explosive mixture was obtained, the condition being adjusted so that the required temperature was reached when the correct quantity of gas had been admitted.

The internal explosions produced under the above conditions, were generally very feeble, and in a number of experiments no internal ignition was obtained.

The current passing in the coil was adjusted by means of a rhoostat and in determining the value to be used for heating purposes, the coil was first brought to red heat, then the resistance was increased until visible light was no longer emitted. The coil must therefore have been at a temperature of over 300°C.

At this temperature prolonged contact with the gaseous mixture, would tend to exidation of the combustible constituents, with production of a non-explosive or feebly explosive mass.

It is therefore evident that gas cannot be admitted to the box while current is passing through the heating coil.

The casing of course never attained the temperature of the coil, and to determine its temperature at anytime during the period of heating, and after the heating current was out off, a number of time-temperature observations were made for different values of heating current. Time-temperature curves were plotted from the observations, and by referring to them it was a simple matter to select a rate of heating and cooling so that the required temperature of casing would be attained at a certain specified time after the heating current was out off. See fig. 24.

Buring the cooling stage gas was admitted to the casing and then to the explosion chamber, the operation being completed when the casing had reached the required temperature.

By adopting this precedure the possibility of exidation of the internal mixture was eliminated, but another undesirable feature was introduced.

As the box apoled, contraction of its gaseous contents

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would occur, and gas would enter from the outside to maintain equilibrium of pressure. It follows therefore that the internal mixture would gradually alter in composition as the casing cooled.

Somewhat similar results may be expected when the external mixture is admitted, before gas is passed into the small box under test.

In this case there will be a mixture containing a small quantity of coal gas, introduced by diffusion and contraction into the casing, before the real explosive mixture is admitted.

We therefore do not know the exact composition of the mixture in the casing, on account of the changes continually taking place, so that while every effort was made to have the internal and external mixtures of same composition it is doubtful if this was ever attained.

The experiments were carried out at critical lengths for gaps from .100 inch to 0.000 inch, so that any change would be readily detected, because the margin of safety is necessarily very small at critical lengths, and any slight alteration in the conditions of experiment, might produce large differences in results.

RESULTS: -

The experiment showed very clearly that the casings remained "flame-proof" until temperatures from $103^{\circ}F$ to $110^{\circ}F$ were reached. At temperatures above $110^{\circ}F$ external ignition could be produced on every occasion until by continual heating the casing had reached a temperature of $155^{\circ}F$ - $160^{\circ}F$, when flame capable of causing internal ignition no longer passed.

Further heating of the casing did not seem to alter the results obtained at 160°F.

CONCLUSIONS: -

It has been pointed out that the higher the initial temperature of the casing, the less will be the cooling when ignition takes place, so that the maximum temperature attained by the explosion will probably be higher than normally. We have also observed that decrease in potential heat of mixture accompanies rise in temperature of the casing, if the latter contains an opening communicating with the outside.

Now, these two factors operate at the same time, the former increasing in power as the temperatures rises above atmos: spheric, while the latter becomes more apparent as temperature increases.

There is therefore every possibility that we may reach a temperature when "temperature increase of explosions" has a more powerful influence then "the reduction in potential heat effect."

When this state of affairs is reached, external ignition will cocur, and will continue until the potential heat of the mixture is reduced to such an extent that the maximum temperature attained (taking into account the increase due to initial temperature) is relatively low.

Under these conditions cooling by contact or expansion, will once more assert itself and thereby prevent external ignition.

In figure 22, percentage loss of explosive mixture is plotted on a base of temperature in ${}^{\text{O}}\text{F}$, the initial temperature being taken as 55 ${}^{\text{O}}\text{F}$, and a barometric pressure of 29.8 ins is Assumed throughout.

From this graph we observe that the percentage loss of mixture is 9.75 per cent at 110° F and 16.74 per cent at 160° F and if the above hypothesis is correct these points probably represent the upper and lower limits of a flame-passing zone.

<u>TABLE</u> Loss of Mixture w Rise in Temperat		CALCULATION OF LOSS. Vo. of Casing is 0.123 cub. ft. Find volume at atmospheric temp 55 F that will cocupy 0.123 cub	. of
Temp. Loss Per in in ^O F. Cub.ft. Lo		at 110°F. $V = \frac{V_{11}}{T_1} - \frac{12}{57}$.1110 cub. .1111 cub. ft. at 55°F and 29.8	0
70 .0035 2.	84	occupies 0.123 cub. ft. at 110° 29.8" Bar.	F and
	12	volume (at atmos. pressure and t that is lost = $0.123 - 0.1110$	
$\begin{array}{cccccccccccccccccccccccccccccccccccc$	10	er Cent. Loss = .012 cub. ft. .123 I c	

170

.0225

18.29

- 41 -

The margin existing between ignition and non-ignition of the external mixture must be very small before the effect described above can be shown. Under the conditions of experiment stated above the effect was clearly observed.

To determine whether the effect was observable when the casing contained explosive firedamp mixtures a quantity of methane was generated from Aluminium Carbide, purified and stored in a gas tank. The necessary amount of methane was drawn from the tank as required.

The results from this series of experiments closely resembled those obtained with the coal gas explosive mixtures, as will be seen from the tabulated results at the end of the thesis.

ANALYSES OF GAS MIXTURES_USED IN THE EXPERIMENTS: -

Samples drawn from the casing were analysed periodically. It is of interest to compare the results of analyses of samples, drawn when the casing was at Stmoepheric temperature, with those drawn after the gas had been in contact with the heated surface for a considerable time.

ANALYSES OF SAMPLES OF GAS DRAWN FROM THE CASING.

<u>No. 1</u> .		(a)	
Barometrio Pressure	30.24"	Sample	45.8 008.
Atmospheric Temperature.	60°F.	K.O.H.	45.4 cos4008002
Temperature of casing when gas was admitted.	60°F.	Br. 4K. O. H	45.3 ccsl ccs. CmHn.
Temperature of casing	60°F.	Руго.	38.05 oce7.25oce. 02
Time between admission of .	.030oub.ft.	<u>RESULT</u> : -	0.87% CO ₂ 0.21% C _m H _n 15.80% O ₂
gas and drawing of sample.			

		(-)		
(b)		(c)	1	
Sample	40.00ccs.	Sample	63.40 0 08.	
KOH	39.66 00s 3400s.002	KOH	62.90 0 05.	1500s002
Вт.+ КОН	39.56 ocs 100030 Hn	Br.+KOH	62 .7 0cca.	.20ceC _m H _n
Comb.	29.77 ocs9.79ccs.Cont	Comb.	47.00003.	15.7cosCont
KOH Pyro	26.99 ocs2.78cos.CO ₂ 26.36 ccs63ccs.O ₂ left.		43.00ccs. 42.00ccs.	4.0008 CO 1.0008. 02 1eft.
02 in Sampl	Le = 6.32 dos.	0 ₂ in Sa	ample =	10.02 dos.
0 ₂ after o	omb. = .63 cca.	02 after	comb. =	1.00 cos.
. 02 used comb		. 02 in 00		9.02 ccs.
H ₂ = Tota	al Contraction - O2 used in comb.	H2 = Tot	tal Contrac used in Co	
H ₂ = 9,7	79 oca 5.69 oca.	$H_2 = 1$	5.70 acs	9.02 dcs.
$H_2 = 4.3$	10 ocs.	H ₂ = 0	6.68 сся.	
OH. = <u>2 x</u>	Cont CO2 formed - H2	СНи = 2	x ContCO	formed - H2
Zeen - I	3		, , , , , , , , , , , , , , , , , , , ,	
CH4 = 2 x	<u>9.79 - 2.78</u> - 4.1	$CH_4 = 2$	>	- 6.68
OH ₄ = 5.6	- 4.1 = 1,5 ccs.	$CH_4 = 9$.13 - 6.68	2.44 009.
co = co ₂	- CH ₄	CO = 0	ю ₂ - сн ₄	
00 = 2.7	8 - 1.5	CO =	4 44	
00 = 1.2	8 008.	C0 =	1.55 000.	
RESULT.	 • • • * * * * * * * 	RESULT.		
6 A 1	0.85% CO2		0.79% CO2	
An	0.25% C _m H _n .		0.31% CmHn	
		1947 - 1	10.53% H2	
8	3.75% CH ₄ 3.20%CO		3.86% CH ₄ 2.44% CO	
8	15.80% 0 ₂		15.80% 0 ₂	
30	65.90% N2		66.274 No	
	00.00	-	00,00	

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<u>No.2.</u>			(a)		
Barometr	ic Pressure	30.54"	Sample	68.50 oca.	
Atmosphe.	ric Temperature.	57°F.	KOH	67.00 008	1.5008.002
	ure of Casing s was <u>admitted</u> .	300°F.	Br.+ KOH	66.80 0c s.	0.2 ccs.C _m H _D .
	ire of casing mple was withdrawn.	147°F.	Pyro. Result:-	57.00 ods.	9.600a.00
	of Coal Gas ad: to casing.	.0295oub.	<u>REJUDI</u> ,-	2.2% CO2	
	seen admission of drawing of sample.	12 mins.		14.49% 0 ₂	
(b)			(0)		
Sample	41.00ocs.		Sample 40	.10ccs.	
KOH	10.100cs 0.900	008.CO2	кон 33	.30008	.0.80ccs.002
Br. +KOH	40.00ccs 0.100	sca.C _n H _n .	Br. 4KOH 39	.20 cos	.0.1000s.Cg
Comb.	30.40008 9.600	cs.Cont.	Comb. 29	. 80acs	
KOH	28.00 0 03 2.400	ca.CO2	KOE 27	.400cs	Cont. .2.40 ccs. CO ₂
Руго.	27.6000s 0.404 le:		Pyro. 27	.10ccs 0	.0.30cos. 2 left.
O ₂ in Sar	nple = 5.98 c		02 in samp	ole =	5.8 oca.
0 ₂ after	comb. = 0.40 (302.	02 after c	emb	0.3 008.
02 us	aed in comb. = 5.58	ccs.	0 ₂ us	ed in comb	- 5.5 oca.
B ₂ Tota	al Contraction - 02	used in comb.	H ₂ = Tota	l Contract used in	ion - 0 ₂ Comb.
¹² 2 = 9.60	0 cos 5.58 ccs.		H ₂ = 9.40	acs 5.	5 009.
H2 = 4.02	2 cca.		H ₂ = 3.90	oca.	
CH ₄ = 2 3	t Total Cont CO2 1 3	formed -n	$CH_4 = 2 x$	Total Cont.	-CO2 formai -H2
сна = 20	<u>19.6 - 2.40</u> -4.02		CH4 = 219	.40 - 2.40	- 3.9
CH4	5.6 - 4.02 = 1.58 0	JC8.	сн ₄ - 5.	46 - 3.9	1.56 cas.
00 - 00	$D_2 - CH_4$		co • co ₂	- CH4	
00 = 2.	.40 - 1.58		co = 2.4	- 1.56	
008	32 ccs.		co = 0.8	4 008.	

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RESULT:	-

E	rs:	16.71:

2.15%	co2		2.00%	c o ⁵
0.26%	C _n ,H _n		0-25%	C _m H _n
9.73%	H ₂		9.75	H ₂
3.82%	CH		3.90%	сн4
1.98%	CO		2.10%	CO
14.49% 0	2		14.49	02
67.57% N	12		67.51%	N2
100.00			100.00	_

COMPARISON OF AVERACE RESULTS OF ANALYSES OF GAS MIXTURES.

I. Bar. 30.24. At. Temp. 60°F.	Bar. 30.54.At Temp. 57° F.
Casing at Atmospheric temp.	Casing at 300°F when gas ad: :mitted. 147°F when sample with: :drawn.
co ₂ 0.83%	co ₂ 2.11%
CmH13 0.25%	0 _m H _n .26%
H ₂ 10.39% 18.09%	H ₂ 9.74% 18.01%
CH4 3.80%	сн ₄ 3.86%
00 2.82%	CO 2.04%
02 15.80%	02 14.49%
N2 66.11% 81.91%	N ₂ 67.50% 81.99%
EXPLOSIVE MIXTURES SONTAINING HYDROGEN Barometric Pressure. 29.42" Atmospheric Temperature.59.0 F. Temperature of Casing o When gas was admitted. 59.0 F.	NETHANE AND TRACES OF (a) Sample 32.8 oca. KOH 32.8 oca. Br.+ KOH 32.8 oca.
Temperature of Casing when sample was with: o idrawn. 59.0 F.	Pyre. 26.93 cos.5.87cc 02 RESULT: -
Time between sidmission of gas and drawing of sample. 1 minute.	17.92% 0 ₂

(b)		(o)
Sample	25.20 OCB.	Sample 21.60ccs.
Comb.	20.50 ecs.	Comb. 17.6000a.
KOH	19.00 ccs 6.2008Tot.Cont	
Pyro.	18.60 ccs 0.4 ccs.021eft	Tot.Con. 2. Pyro. 16.00ccs 0.3 ccs. 0.2 left.
0 ₂ pres	ent in sample = 4.51ccs.	O_2 in sample = 3.82 ccs.
0_2 afte	r combustion. = 0.400cs.	O2 after com: :bustion30 ccs.
02 ; bua	used in com; tion. = 4.11 ccs.	-
COp f	= total contraction due to stion and absorption of ormed. = Oxygen used in combustion. = $\frac{A}{3}$ T - 20	$H_{2} = \frac{4}{3}T - 20$ $H_{2} = \frac{4}{3} \times \frac{5 \cdot 3}{1} = 2 \times 3 \cdot 5^{2}$ $H = \frac{21 \cdot 2}{3} - 7 \cdot 9^{4} = 7 \cdot 06 - 7 \cdot 06 - 7 \cdot 04$ $H_{2} = 0.04 \cos 8$
H ₂	$= 0 - \frac{1}{3}T$ $= \frac{4}{3} \times \frac{5 \cdot 2}{1} - \frac{8 \cdot 22}{8 \cdot 26} - \frac{8 \cdot 22}{8 \cdot 26}$ $= .04 \ \cos .$	$CH_4 = 0 = \frac{1}{3}T$ $CH_4 = 3.52 - \frac{1}{3} = 3.52 - 1.76$ $OH_4 = 1.76 \cos \theta$
CH4	$= 4.11 - \frac{6.2}{3} = 4.11 - 2.06$	0.09% N2
он4	= 2.05 ccs.	8.14% CH4 17.92% O2 73.85% N2
RESULT:	0.16% Н ₂ 17.92%0 ₂ 8.13% СН ₄ 73.79% №2	
2.	An and the test part that	(a)
Baromet	rio Pressure 29.34"	Sample 34.2 ccs.
Atmosph	eric Temperature 58.0°F.	Pyro. 28.4 ocs. 5.800s.02
Tempera gas wa	ture of casing when a admitted. 300°F.	Result 17.0% 02
Tempera	ture of casing when was withdrawn. 182°F.	(b) Sample 25.0 cos. Pyro. 20.7 cos. 4.3 cos. 02
	tween admission of d drawing of sample 9 mins.	Result 17.2% 02
-		Average Result 17.1% 02

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(0)	(d)
Sample 29.4 ocs.	Sample 54.4 cos.
Comb. 24.2 ccs.	Comb. 52.8 acs.
KOH 22.0 ccs. 7.4ccs.Tot.Cont.	KOH 48.25 ocs.16.15ocs.Tot. Con.
Pyro. 21.9 ccs. 0.1 cos.02 left.	Pyro. 48.05 ccs. 0.2 ccs. 02 left.
0 ₂ present in Sample 5.02 ccs.	O2 present in Sample 10.940cs.
O2 left after comb. 0.10 cos.	O2 left after comb2000s.
0 ₂ used in comb. 4.92 cos.	0 ₂ used in comb. 10.740cs.
$H_2 = 4 T - 200$	$H_2 = \frac{4}{5}T - 20$
$H_2 = 4 T - 20$ $H_2 = \frac{4}{3} \times \frac{7 \cdot 4}{1} - 2 \times 4.92$	-
$H_2 = \frac{29.6}{3} - 9.84 = 9.86 - 9.84$	$H_2 = 4 \times 16.15 = 2 \times 10.74$
$H_2 = .02 \text{ ocs.}$	$= \frac{64.6}{3} = 21.48$ $H_2 = 21.55 = 21.40 = .05 \text{ org.}$
$CH_{4} = 0 - \frac{1}{3} T$	$CH_A = 0 - 1$
· · · · · ·	CH4 = 10.74 -16.15
$CH_4 = 4.92 - \frac{7.4}{3} = 4.92 - 2.46$	$CH_4 = 10.74 - \frac{16.15}{3}$ = 10.74 - 5.38
$CR_4 = 2.46 \ cos.$	CH ₄ = 4.36
<u>Result</u> :- 0.08% H2	Reault: - 0.07%H2
8.36% CH4	8.32% CH4
17.10% 02	17.10% 02
74.46%N2	74.51% N2
COMPARISON OF	AVERACE RESULTS.
I. Bar 29.42. At temp.59°F.	Bar. 29.34. At Temp. 58.0°F.
Casing at Atmospheric Temp.	Casing at 300°F when gas ad: :mitted. Casing at 182°F when Sample
arminen y entre seu trata	drawn.
H2 0.12%	H ₂ = 0.08%
CH4 = 8.14%	CH4 = 8.34%
02 = 17.92%	02 = 17.10%
N2 = 73.82%	N2 =74.48%
44508 54.4201 Alexin, 423, 01011 - 42030	are poster an arrite a
	A CONTRACTOR OF A CONTRACTOR O

68.75

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Where heating is primarily due to some electrica, defect, very high temperatures may be reached and result in destruction of insulation etc.

If temperature of 300 C, are reached in the presence of an explosive mixture of gases, prolonged contact with the heated source will tend to oxidation of the combustible constituents with probable production of a non-explosive mass.

With higher temperatures the rate of heat production due to exidation, may be greater than the rate of heat dissipation, and if this abnormal state of affairs be allowed to continue in: iflammation of the gas mixture will occur.

However, the small, smoke, etc., which must be produced under such circumstances, will direct attention to the cause of the disturbance and should result in closing down of machinery etc., until the defect is remedied.

PERMISSIBLE AMOUNT OF WEAR AT BEARINGS: -

With "protected" motors we have to consider the possibil: ity of passage of flame between the shaft and bearing, in event of excessive wear.

From consideration of results already obtained we are of the opinion that with plain bearings or ball bearings, the length of bearing necessary to support the shaft and armature or rotor, is more than actually required to prevent passage of flame.

The wear of bearing and shaft will produce a crescent shaped gap, the maximum depth of which will be the difference in diameters of the bearing and shaft.

If the length pf shaft were eay, 4 inches, the gap necessary for passage of flame along the shaft, to produce external igniton would be from 3/16th." to 1".

In actual practice such wear would never be permitted, since serious electrical distrubances or magnetic unbalance etc., Would be produced, and necessitate renewal or repair of bearing.



BEARING, AND SHAFT NOT GAD BETWEEN SHAFT AND BEARING, SUCH AS MINHT BE FORMED BYEKCESSYS HOME Experiments were carried out on a box of volume 2.35 cub. ft., fitted with machined flanges 1.375 inches broad. The box was made of cast iron.

A white-metal bearing was cast into an opening in the side of the casing, the dimensions of the bearing being 2 ins. internal diameter, and 2.5 ins.external diameter, and 2.5 ins.long.

The steel shaft which fitted into this bearing, was 7 ins. long and projected 2 ins. from the bearing,

A groove 0.125 ins. deep was cut in the shaft at the inside end of the bearing, and in this groove four stude were set, to prevent the shaft from being forced out during an internal explosion.

After an explosion in a partially closed vessel such as the above, condensation and ocoling occur, and pressure falls below atmospheric.

It follows therefore that the shaft would be drawn into the casing after an explosion, if some means were not adopted to prevent such an cocurrence.

A sorew-stud was therefore set in the shaft at the cutside of the bearing, and had the desired effect.

The arrangement is clearly shown in fig. 25. One end of the casing was tapped to receive a gas tap and a plug for carrying the igniting current cable, and a gap 0.01 ins. in depth was left at the flanges.

The procedure was to admit an explosive mixture of coal and air to the casing and ignite it by blowing a "tin" fuse.

The length of fuse used was 1.5 ins. (No. 24. S.W.G.), the igniting current being 18 amperes at 250 Volts D.C.

In the initial experiments the bearing and shaft were * "tight fit", but as the work proceeded, the shaft was gradually reduced in diameter by predetermined amounts, strict observation

한 것은 것 같아요. 그는 것 같아요. 이렇게 집에서 가지 않아요. 나는 것이 아무렇게 한 것이라요. 이는 것

the let dailone acted to the differences

being kept to ascertain when flame first appeared at the bearing.

At each reduction in diameter of shaft, a number of tests were carried out in the "open", followed immediately by gimilar tests with the casing surrounded by an explosive atmosphere.

In none of the latter experiments was it possible to produce external ignition.

Occasionally when 'proving' the external mixture after an internal explosion, it was possible to see the flame being drawn into the casing, between the flanges and also at the bear: :ing, the action producing a whistling sound.

In a large number of the tests the screw stud was withdrawn to see whether ignition of gas would occur when the external mixture entered the casing in the wake of the shaft, after the internal explosion.

Rapid cooling must occur since ignition of the external mixture could not be produced in this manner, but often when proving the external mixture a slight explosion could be heard within the casing, proving the existence of an explosive mixture which must have entered when the shaft was "drawn" in.

white emanation was discernible at the bearing during ignition.

As the shaft was reduced in diameter, this "ëmanation" gradually gave place to distinct brownish flames, with wisps of blue and a few "sparks", then as the gap became greater bright red flames and brilliant sparks were observed during ignition.

When a shaft diameter of 1.945 ins. (giving a gap of 0.055 ins.) was reached reddish-blue flames extending to the end of the shaft were apparent, and as the flames increased in luminosity the the white emanation was no longer visible.

After experimenting with the shaft dry, a film of oil was introduced. The immediate effect was considerable shorten: ing of the flame, and in many experiments complete quenching of flame resulted.

Considerable quantities of smoke were expelled during

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the explosion, and the shaft on the outside of the bearing was covered with globules of oil.

The burning of oil between the shaft and the bearing would account for the volumes of smoke at the bearing, while the globules of oil found on the shaft would be deposited there from spray of oil ejected by the gases from the explosion.

After this observation, it might be suggested that the white emanation spoken of above, was merely products of combustion expelled by the explosion. On examining the side of the "explosion chamber" a deposit of grey matter was found directly opposite the bearing indicating that the above view is probably correct.

The fifth experiment with the shaft reduced to 1.945 inches ended disastrously. The casing burst under the strain of the explosion and damaged the explosion chamber.

Part of the casing containing the shaft was carried shrough the end of the chamber and came to rest 10 feet from it.

On examination it was found that the casing had been sheared along the junction of the sides and bottom.

The gap at the bearing (0.055 ins) would never have been allowed to form in an actual machine, since disturbances etc., already mentioned would have necessitated renewal and repair before the wear had become so marked.

The results obtained are shown tabulated at the end of the thesis.

A smaller box made of cast steel, was now produced and adapted for experimental purposes in the manner already described. The volume of this box was approximately 0.123 cub. ft., with flanges 1 inch broad. The bearing was of brass and of the following dimensions, length 1.625 ins., internal diameter 0.625 inch, external diameter 0.875 inch.

The shaft was made of mild steel, and was 4 inches long, and had a diameter of 0.625 inch.

The shaft and bearing were intended to represent, a

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controller shaft and bearing.

In the first series of experiments carried out on this box the shaft and bearing were a "tight fit."

The bearing was increased in internal diameter by degrees until flame capable of causing ignition finally passed.

Sparks and flame were first observed with a gap of 0.047 in. At the bearing, the flame being blue in colour and extending and extending only a short distance.

As the gap at the bearing was increased, the flame from the explosions increased in length, and changed in colour from blue to red, while at very large gaps of the order of 0.10 inch the flame at the gap was a vivid white colour.

External ignition was ultimately produced when the difference in diameter between the shaft and bearing was 0.125 ins.

The flame produced in this case extended approximately 3 inches beyond the bearing.

Such excessive wear would never be permitted in a controller shaft and bearing, since faulty contact of studs, fingers etc., would be produced giving endless trouble, and necessitating attention before a gap of such dimensions had been formed. The results will be found tabulated at the end. <u>CONCLUSIONS:-</u>

Although the first series of experiments could not be completed, sufficient data was obtained to indicate that in general the length of bearing required to support the shaft etc., in more than sufficient to prevent passage of flame between the shaft and bearing.

The danger of passage of flame need not be the deciding factor in determining the permissible amount of wear.

The factor of greatest importance is the prevention of electrical or magnetic disturbances, and if strict attention be directed to this phase of the question, no fear need be enter: stained regarding the possibility of ignition of gas by leakage at

- 52 -

the bearing.

It was found possible to complete the second series of experiments and the results are an interesting addition to those of the first series. The production of faulty contact, due to excessive wear at the bearing, has been mentioned. Aroing thereby produced would increase the possibility of ignition of explosive gas mixtures present in the controller casing. It follows that if protected the casing was not adequately/external ignition might be produced, but the likelihood of passage of flame between the shaft and bearing is remote, since it is hardly conceivable that apparatus would be allowed to fall into such a state of disrepair that a difference of 0.125 inch in diameters of shaft and bearing existed.

We may conclude therefore that if beasings are renewed when trouble in making contact is experienced, there will be no danger of passage of flame between the shaft and beasing, capable of igniting an external mixture.

GENERAL SUMMARY OF RESULTS AND CONCLUSIONS: -

Cotton gaskets do not constitute a permanent flame proof joint, because they deteriorate rapidly in humid atmosphere.

Flanges 0.5 inch broad do not adequately protect a casing. They do not prevent passage of flame capable of causing ignition of external gas mixture, unless they are machined and in metallic contact. Even under these conditions the margin of safety is small.

An adequately protected casing, maybe rendered unsafe by attaching an inadequately protected piece of apparatus to it. All apparatus should be protected to the same degree.

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Where flanges are internal (i.e. extend inwards from the side of the casing), all stud holes should be bottomed, since omission of a stud would provide direct communication with the outside atmosphere.

Ignition may be produced by the operation of everload releases.

Cooling of gases from an internal explosion may be effected by either of two methods, or a combination of these two methods. The two Methods are (a) Cooling by contact with a large metallic surface. (b) Cooling by expansion through a small orifice. Cooling by expansion predominates where the release area is small, consequently pressures developed by the explosion of gas mixtures are relatively high.

Cooling by contact cocurs when a large release area is provided. In this case high pressure develop: ;ment is prevented.

Where a gap is left all round the perimeter of an opening which is protected by broad flanges, large cooling surface and large release area are combined. This state of affairs satisfies the definition of the "flame-proof casing definition of the B.E.S.A., since cooling of flame is adequate and pressures developed are low.

Devices which provide a large cooling surface and a small release area do not completely satisfy the requirements.

There is a distinct "flame-passing range" in gas mixtures, and for a given depth and length of gap between flanges of given breadth, the higher the value of the igniting current (till 40 amps. is reached) the

and the effective of the state of the

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more nearly does the limits of this range approach the limits of inflammability of the mixture.

The phenomenon is observable when the "margin of safety" is small.

Coal dust present in switch gear casings will probably interfere with the electrical continuity, and in event of an explosion of gas occurring in the casing, will tend to produce slightly higher pressures, but the flame-produced is not so intense as from an explosion of gas alone.

If the correct proportions of coal dust and gas are present very high pressures will be developed.

Increase of value of igniting ourrent has no effect on ultimate pressure developed, but probably affects the rate of pressure development.

Pressure developed is dependent upon the release area provided. Where a large release area is provided a large mass of gas is expelled unburned when ignition occurs. This tends to reduce temperature and pressure attained, since these depend on amount of gas consumed.

<u>WEBBED COVERS:</u> The provision of internal webs in casings is worth considering. These in addition to strengthening the casing increase the cooling surface presented to the flame when ignition occurs. This tends to lower the temperatures attained.

HEATED CASINGS:- If casings are adequately protected (i.e. have a good margin of safety) no doubt need be entertained regarding the ability of protective measures to function properly at temperatures considerably in excess of that of the surrounding atmosphere.

Increase in temperature of the casing, whilst reducing the efficiency of the cooling surface also

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causes reduction of nett weight of gas contained in the casing.

This effect more than compensates for loss in efficiency of protective measures.

<u>PERMISSIBLE WEAR OF BEARINGS</u>:- With plain bearings the length required to support the load is more than necessary to prevent passage of flame along the shaft through a gap formed by reasonable wear between shaft and bearing.

Before conditions are such that flame capable of causing external ignition can pass, the electrical and magnetic disturbances will be of such a magnitude that the apparatus will have to be closed down for renewal of bearings, etc.

With controller shaft bearings the length should be from 2-3 times the diameter of the shaft, and renewals should be made when the amount of wear is such that trouble is experienced in making contact.

In each case, if the remainder of the apparatus is adequately protected there will be no danger of external ignition.

RECOMMENDATIONS: -

Broad flanges with a gap between them should be provided at all joints.

In deciding the depth of gap to be left between flanges the volume of the casing, the perimeter of the opening and the breadth of flange should all be considered..

Figs. 9 and 10 have been drawn from results obtained by experiment, and by referring to the curves the maximum depth of gap and necessary breadth of flange will be found for volumes and perimeter within the limits plotted.

The gap may be provided in a variety of ways, e.g. by means of spring washers, or study projecting from the surface of the casing flange, but an allowance should be made for the at the ends of covers and casings, otherwise the gaps at the ends may be considerably in excess of the safe perimeter gap.

Cast steel casings are preferable to cast-iron casings, unless strengthening webs are provided in the latter.

In the experiments described in the foregoing pages, and in the interpretation of results etc., the practical aspect of the question has been considered of primary importance.

The conclusions arrived at and the recommendations and suggestions advanced are therefore made on this basis.

At all times every effort was made to reproduce a practical atmosphere, and have conditions resembling as closely as possible, those met with daily in mines.

The author desires to place on record his indebtedness to Professor Burns and Professor Parker-Smith of the Royal Technical College, Glasgow, under whose direction he worked. The kindly guidance and advice of these two gentlemen were much appreciated.

- 57 -

- 50-

									90	
-	C 14	Gas IN	16AMIN 6	BRURDLE	GA	ō	Inner			7
10	GAS METER	CUBIC Fr.		OF	94	-	IGNIT	TON		
No	READING	a alan	IN	FLANGE	LENGTH	Depru	le ra que	Errent	REMARKS	
		E.C. 5.B.	AMPERES		Incers	INCHER				
1	94167-94-56 <u>6</u> 941619	.043	- 29	1	-5	- 025	Yes	V IES	VIOLENT EXPLOSION RINCE BOLT SHEALED COVER TOLIST AND ALLOW	e
-	10 411- 98 65	-024		-4			1/		FATERNAL MARTINE EXPLOSE	VE
2	01-05	-556	- 6°		•	-	YES	No	Palentar - Harvie	-
3	100-111		-		-	-	Yes	No	1 0.	
-	100-6 64	the second se							and the second se	
4	140.927	-413		_		- 11	Yes	No	Do	
5	101-427		-			14	YEL	No	U. Do	
17	1-406	·059					44			
6	1-9/9				-	-	Yes	No	- Do	
7	1.965	And in case of the local division of the loc	100			10	Yes	No	Do Do	
8	2-665	044				-620	Yes	No		
_	5457	-035	-		375	020		140	'e Do	
q	4.842	and the second s	-	*		-	IES	No.	D_0 D_0	
10	5-205	110		-	15.6	-	Yes	No	Do Do	
	5.281		-		1					
H	5.726		-				Na	No	FAR CONNECTION BROKEN	
12	5.765 6.055	CONTRACTOR OF THE OWNER.			.5	1054	YES	YES	THREAD OF WING NUT STRIPPED CAUSING SOVER TO LIFT	
-	6-062	201	-	- -			1.1.1	1	EATERNAL MILTURE EXPLOSI	νe
13	6.293		-		<u> </u>		YES	N ₀		
	6-336	-310					YES	No	D _o D _o	
15	6.6.63	- 0.34				- 10	Yes	No	Da Da	
-	7.025			<u>+</u> → —				No		
16	7-474	and the second second second		L			\rangle_{ES}	N_D	Do Do	
17	7-15+-7896	- 637	L		-375	. 472	Yes	Ne	Do Do	
1.00	1-597-2-641	-200			-10	- 14			<i>D0D</i>	
18	4 964	247		L		-	YES	Νø	Do Do	
19	7-0-16	1256 -224			-50	-/aq.	Yes	No	Do Do	
	11-309-10-35	-06	3	+ $-$		104	- /	-		
20	11-309-15-25	306	· ·		1	-	YES	No	Da Da	
21	10.70	1 2446					YES	Yes	WADD EXPLOSION CHANNEL USED	
0.	13 726	1 1 1 1 1 1 1 1							VOLUME 16-6 CUBIC FEET	
22	16 571	2-45		+ -		~	Yes	YES	D_o D_o	
23	16-6 34	9 3.301	141			1046	YES	YES	DISTINCT EXPLOSIONS	
	1111013/0201	1 03		+	1000				EXTERNAL MINTURE EXPLOS	
24	16761	7 2 4:7			.625	.026	Yes	No		
22	31-41	3444	H .			.100	Ses	No	D_o D_o	
16	39.24 - 23.40	4.34	-		1.90	073	Yes	No	Do Do	-
	15 10	3344		+ -	.75	-13		-		
27	19. ze	6 190		L -	1		Yes	N _D	Do Do	
28	20.74	1 -01; 1312	E -		1.0	.080	Yes	YES		
	19611-1253	2 03/	1	+ -	1.0					
22	46 36	1 3 4 7		+ -			Yes	YES		
130	40.24	5 2 440	4 .			100	Yes	No	EXTERNAL MINTURE ENPLOST	¥Æ.
130 31	51 9 62 - 55 10	2 103	2 00				1.1	Yes	Two DISTINCT EXPLOSIONS	
1.000		0 = 0 + 0 (- 11 - 11 - 11 - 11 - 11 - 11 - 11 -	25	+ -		.080	Yes	-		
32	62.61	7 592/	7 -			. 10	Yes	YES	EXTERNAL MILTURE EXPLOSI	vε
đ,	- 65240-6592	0 -26	20		1	-040	YES	No	Do Do	
34	64 1	1 2172 -03	0	9	+		-			\rightarrow
		2 19/19		+ '	-	+1	Yes	No	Do Do	
53	75.41	7 3 297	- +		-	1	YES	No	De De	
30	5 22130-2214	-03	6 10		-375	+080				-
		3212	14	-	-15		Yes			\rightarrow
3		4. 307			-625	-	Yes	N,	De Do	
34	A 10-045-17-5	10 11	5	+	+475		Yes	No	Do Do	
30	7 - 14-01- 4/6	10 2066	1	-	10			1.1		
	M	1 2 4 2			12		Yes	No		
4			7				N.	N	FUSE CONNECTION BROKEN	v
4	98.0	6 3:251 70 103	2			-		1000		
	101-11	5 345		+		1960	Yes			$ \rightarrow $
4	- 6 11	4	<u> </u>	1.7	1-75	1058	Yes	Yes		
4	5	-03				1	1.1			IVE
4	4 374	2 2428					Yes	-		_
1.17	11.0.6	7 1111			1	-	YES	Yes	Two DISTINCT EXPLOSION	/S
- Ĥ	5 11.10	2 192	§ +	- 6	11			Yes		
4	6	1 3-24%	0	-						
	- 761	1 1 V 46	1		1 -	1	JE S	Na	ETTERNAL MISTURE EXPLOS	VE.

N=	Gas Meter Reading	Culasi	ё . , S В	Current in Amperes	Brienin of Flaure		0.000	JG N I T		REMARKS
47	17632-17660 11-011	3 342	037	12	1"	1-75	-056	Yes	No	INTERNAL MISTURG TOO RIEN ESTECHAL " EXPLOSINE
48	10029 21.354	3 547	0.81		4	4		Kez	Yes	
29	14191-14-44 - 1718-	3745	ind i	- 4	- 1 -	-		755	Yes	
12	#7 633	Street and	• 0.3 <i>e</i>		+	1.5		Yes	Yes	
51	10 707 11 94 l	3-217	976	i tan		1条	1	Yes	Na	EXTERNAL MISTURE EXPLOSIV.
52	n 1)		731		*	1.375	~	Yes	Yes	THE DISTINCT EXPLOSIONS
53	37.317 1: 192-je4.50		030		1		-	Yes	No	EITERNAL BAPLOJIVE
54	-++++7- 63-944		431			~	0.441	۷ _{€J}	Yes	Two DISTINCT EXPLOSIONS
55	17474 - 41414	1.111	-030	~		- 11	~	Yes	Yes	
56	1771 - 52 73	2756		t	-	1.25	-	Yes	No	EXTERNAL MIXTERE EXPLOSIVE
57	N 740-52 JJ 5	3 (2) 2	0.24	6	-	•	-	Yes	No	Ø. O.
58	57 649-60-811	1 1024	+#30	-	-	1.75	.020	Yes	Ne	0° Da Do
59	60-374 - 6-8-314 - 8-374 - 9-9-9-9	3.256	DLE	· ·		2.25	040	Yer	No	Pa Da Da
60	• 3-6 3.∩•63.⊧€• }a6	3.791	• 0 3 0		+	-		Yes	No	D. 0, 0,
11		12231	-30		L	275	- 10	Ver	No	D. C. D.
62		3711	•030		-	55	-1-	Yes	A/a	Do De P
1.1	71.76 -51.79 71.76 -51.79	2.241				40		Yes	N¢	Do Do Do
64		3 22 4	·027			× .5	038	Yes	No	Do Do Du
65	26-31 26-31	ान्छ 😳	-030			175		Yej	Ag	0n 0: 0.
66	17.60	3.270		-	÷	6.5	- 04,0	Yes	No	D. D. D.
47	90.300	1 1 1 20	-030	1	1.12	6.5	04 2	Yes	No	D. D. D.
68		1 7	-01 -03			6.5	- 20	Yes	Nυ	$D_e = D_e = D_O$
69	97.641	1 3 3 70		1-		160	038	YES	YES	VINLENT EX
71		1000	130		1	10.73		Yes	Yes	
71	-	0 3270		-	-	\$ 75		Yes	Yes	
7		1 3761	-120	- ·	1 -	9.0	-	Yes	Nº.	FEEBLE INTERNAL EXPLOSION EXTERMAL MATTURE EXPLOSIVE
17	10.90	12/2			-	9-0	-	YES	YES	VIONENT EXPLOSION
74	1. 16-10) 18-48410-11	3 3 2 AL				fé	-	Ves	100	Two DIATINGT EXPLANIONS
73	2015	5 321		-	· ·	7.0	-045] Yes	Yes	
-10		1 2 113	1	۹ <u>.</u>	1-	6.75	-) jes	Year	
- He	50.11	10 3-16 8	1.03	1		6.0	-	Ves	Yes	
17	29-141-32	11 3-740		-	-	5-75	-		Yes	والمستعملين والمستعمل
15	1 12-44 25	16 2211	-03	1		5.2.5			s Ve s	
E.	1 10/172-003	64 1.747 64	+0.5		+	F-12.5	-	Yei	1000	Two Distinct Expensions External Mictore Expensions
Ē	650	28 2-241	03		-	4:00		1 <i>E</i> J V	1.1	Bares Know Acres Co. Take Arres
1	- H.T.	24 3 174	0.31	_	+	2.123	1.1.1	Yeu		BATERNAL MUTURE
1	- 72 6	TABLES	-03	7	-	29	1.021	Yes L	1	EXPLISIVE
		10 3-74-9	1.03		+			Yes	-	₿ ₀ D _a
	99.4	12 3 46 1	-03		+	10	4	YEI		Di De
	7 - 1121	100 J -16	8 -0.3		+*			} I € S		De
ĺ	7 - 57	20 T 110			1+1	+	035	$\frac{Y_{ES}}{v}$		
3	9 - 9.6	72 3 -20 64	1 100		-		030	Yes		
9	15.14	87 (\$194) 12	- 77		- 4.		030	1.1	Nø	Do D,
1	13.21	1.2	-03	la .			030	YES		D, D,
T	- 1108 · - 1108 ·	1.70	- 10				-) les	∧/₀	$D_a = D_c$ $D_c = D_a$
	1 18 16	9	1	7		<u> </u>	·030	1.16	c M ₂	$\rho_e = \rho_{e_{\pm}}$

-5 9-

-	- 60-													
	GAS METER	Gas	In	Contrar	B Knara	GA		lann	TI PN					
No	READING	EC	SB	Annas	FLANCE	LEND+H IN Increa	Depte W	/ee	-	REMAR	45			
93	27.406		3/6	10		-5	-700	Yes	No	EXTERNAL M	ATURE EXPLOSIVE			
94	1.7.2	2-100	3/2			.625	100	Yes	and the local division of the local division	Do	Do			
95	30+926	2.900	-57			46	.100	Yes	No	De	Do			
96	36-156	2.900	日花	la:		.625	.100	Yes	No	Do	Dr			
97	27:26.4 AU:26.4		B-17-8			·875	"	YES	No	Do	De			
98	40-364		316		-1	-	ICP.)es	No	Do	De			
99	43 475 44.37F		-31/4		-	1125		Yes	Yes	-				
100	- 10-59A	19 - 200	-320		•		100	Yes	Yes					
101	14440-20 009 53-613)株正// 9	1404		. e-	1.0	•	Yes	No	EXTERNAL I	MITTORE EXPLOSIVE			
102	1 53 ž 26 7 4	2 900		1 *				Yes	No	Da.	Da.			
103	Lizizka	1-102	312	1		2.25	.070	$\dot{\gamma}_{\rm ES}$	No	Do	Do			
104		1.900	3/2	· ·		2.375	4	Yes	No	Do	De			
105	the second se	2.90		1		-		Yes	No	Do	De			
106	62.94	2.9M	1.517	1 million 1		2.75		Yes	Ne	Da	Do			
107	11002-11-20 16-20 46-34	12010	300		-	2.95		Yes	Ne	P .	p.			
143		15.70			-	2-625		Yes	No	De	₽ _v			
109	92.00	2700	300	1		2.875	*	Yes	No	Da	Du			
110		2.70	-300			3.00		Yes	Yes					
100		1 2 700			-	3.00		Tes	,	M	E al com			
112	10/012	2.700	030	6	-	6.00	092	YE+			XTORE EXPLOSIVE			
113	A 812 4-3/2	2.700				7.0"	1052	Yes		00	De			
114	7-012	\$-764	131			7.5			No	D ₄	O _n			
1/5	10.754	2.000	108		- v.	8.0	-1-		Yes	Fire que				
116	3.6A	2 2 Hoy 4	0 • 3/ 2			7.75	-	VES	Ne		MIXTURE EXPLOSIVE			
117	16-580-164	2-10	-312		**	7.875		Yes	/No	Do	D ₀			
115	19 10	AF # 7/	e -312		÷	7-475		Yrs V	N.	D. D.	De De			
12		71		-	01	800		Yes Yes	No	D _a	D.			
121	76 M		1.27.2		1.	825		Yes	No. Yes		TEXPLOSIONS.			
12	a \$9.113-394	2 7 15	- 37	_	1-	1.0		Yes	_		and the second second			
(2		71 612-0	1.3/	2	1.	8.50		Yes						
12	35.64	2 2 2 2-35	31	2 .	-	8.75	-	Yes		Do	ū,			
12	< 19 044		0/3	L .	-	9.00	+	Yes	+		м. — — — — — — — — — — — — — — — — — — —			
12	45.07	11 10 \$3	1911	4 .		9.25		Yes	t	Da	Do			
12	I = 927-472 50 pt	59) 14 - 3 4 5	0912	- · ·		9.50		Yes	No		D ₀			
- 22	F 31093-504	20	10 272	_			-	Yes		Do	Du			
	10 33.5	(4) 1 245 7	• 20		н	- 10	10	Yes	No	Do	Do			
1		17 8/07	10 I	-	1.4	10425	v	Yes			DIEWN OLI			
	21 2800 - 24	22 72 2 3 1	31		1.			Yes	No		PTURE EXPLOSIVE			
	57 k229542 629	Co. C. E1	101	-	Ŧ	13.5	- 11	¥es.	Na	De	0 ₀			
	13 -2 - 2 - 2 - 2	22 2	10		- 10-		-	γ _E s	No	Po	De			
	14 72.2 72.2	17.54	<u>का</u> इस			14.5	6	Yes		Do WEDDEN PLA	Do BURNED OUT			
	15 Tints 77 15 7 16 7 16 7	45 8-7	12/ 10 1/2		18	eet = a	+	YEJ	+					
	74-3 37 7122-744	20 27				"		Yes	-		TITURE EXPLOSIONS			
Ę	78 N.S	1.124	10 10 130		0	11.5	1038	Ves V	-	10000	T EXPLOSIONS.			
	RI	10 2.71	u e		1	k€ 5	1052	Yes	Yes	We DISTINE	TYPED 370 W3			

No	GAS METER Reading	C obie		CURRENT	BREADTH OF FLANGE	G A Lén 6 7 h	DEPTH	IGNI	TION External	REMARKS.				
	81.312.51.462	EC	·320			INCHES	<u>[N CHES</u>							
12.4	87-362	2.700	-312	15	1375	18.5	.052	YES	YES					
140	67366-67-676 an 941-	2 -1 AM			~	16		YES	No	EXTERNAL MILTURE EXPLOSIVE				
141	?^&&&q ?	17 700	*313 (÷	Yes	les	GAS PIPE PLUG BLOWN OUT				
142	96417		-3/3		16	-		YES	YES	TWO DISTINCT EXPLOSIONS				
14.3	46 77A 99 416	1000	315	-	н	15	÷.,	YES	No	EXTERNAL MISTURE EXPLOSIVE				
144	99726		-3/2	-		۴.,	÷)es	No	Do Do Do				
145	9736 5438	· · · · ·	-3/2			16	4	Yes	No	D. D.				
156	3 730		-3/2_			16.75		YES	Yes					
147	\$ 762		-3/8	-	4	27	·04	Yes	No	EXTERNAL MILTURE EXPLOSIVE				
144	11-2.62		-112			ALL		Yea	Xes					
	4 551 14-51 A		-99.9			ROUND 22	·037		No	EXTERNAL MIXTURE EXPLOSIVE.				
44	17-024		-3/2		-			Yes		2				
150	20-396 20-596 - 2090	2700	·#/k			27		Yes V-	No	Do Do				
151	711-59 + - 71-914 	2 700	-312			30		Yes	No	Da Da				
102	1. 14	2700				40	-	Y∈s	No	Do De				
123	30.24	2-20	-307			Rasep	1036	Yes	No	Do Do				
152	37.970	<u> </u>	1000		-	-tr	~	Yes	Ne	Da Da				
125	15212	2.00	3/2			4		Yes	No	D _e D _e				
156	54 263 -3460 62-20	21.00	-315	•	1.75	10	.08	Yes	No	Da Do				
157		1 2 1 . 140	-350			2.0		YES	Yes	SHORT DISTANCE PISCES BLOWN OUT FROM BETWIN N FLANGES				
154	45.52	2400	350			2.75	-	Yes	No	EXTERNAL MINTURE EXPLOSIVE				
159	48473		.350			3.5		Yes	No	D _a D _a				
160	51.641-31-96	2404	D.FD	-	111	3.5		YES	No	Do Do				
141	54 641	2	350		1.	4.25		YES	No	Do Do				
162	57.04	7600 / / 7600	.350			6.25		Yes		Do Do				
163	66.34	4	1358						_					
	67-94	/ 24-05	350		-	6:25	-	Yes	No	Do Do				
144	45.99	F 146.00	.350		17	7.25			Yes					
	HE RO	2 601				6.50	-	Yea		EXTERNAL MITURE EXPLOSIVE				
140	7/- 4/	240	.350	-		6.25		Yes	No					
16	76 76	9,2400		-	-	ADUND		No	ND	FULF GENNEET = A FROND BRONDA				
16	7775	1 2 6 6	-354			10		Yes	Na	External Marura Exercision				
14		1 21641			-	4	046	Yes	YES					
12		19.4	- 750				· 041	Yes	Yes	TWO DISTINGT EXPLOSIONS,				
12		1.566	.350		-		-039	Yes	No	EVERANAL MOTIONE EXPERSIVE				
17	7 10 2	1 24.0	-350		-	11	.058	Yes	No	Do De				
17	3 99901-902	9/1 17 1 244 10	• 3,570			12	-11	Yes	Yes	in the second				
17	43 40 	1/ 18 17-10	1 351	2 -	-	11-5		Yei	No	GATRENAL MINY ORE EXPLOSIVE				
12	2 場	16	35	ā		12.0		Yes	No	Do Do				
	L 44.762-92	11 11 = 1 m	- 551			4	-	Yes	No	Do Do				
12	9 20	10 - 10 10 - 10 10 - 10	1.11	_	-	14.25	4,	Yes	No	Do Do				
17	1 441- PG	4	2.8			1.		Ver	Ka	TWO DISTINCT EXPLOSIONS				
1	0 7-245-75	4 7 m	•25		-				Ver	D, D,				
	10 10105-017	1 2 D	1.26			10100		Yes Yes	Yes					
	- 40	4 741 14	e - 13	a		12-625			185	Eardithe Missier Expansive				
	17 160	94 2 .40 8	.241		~	12.125	-	Yes V						
	47	10 2.60	.24			-		Yes	No	$D_o D_e$				
100	F. 21.4	6. 2m	el			12-625	-	Yes	her					
	116	241	125	74		15.125	•	Yes	YES					

	-62-												
No	GAL MATER	Gas cubic E.C. I	Seet	Current In	BREADTH OF	G A Length Inches	P Deptu	IGNIT		REMA	R K5		
185	1507-2601-7 96 56 7		- 10	15	1.75	15-125	-045	No	No	FALL Not PR.	FALLY COMMETED		
186	16-477-26-7/7		•240	1.			.050	YES	No	ELTERAN AL MIST	URE EXALOSIVE		
IAY .	18:95 7 30:457	20.02	.240	4		14.125	·048	Yes	Yes	Two Disting	T EXPLOSIONS		
188	31-14-7	2.000	-940				-4	Yes	YES	Do	De		
1841	35-//3-7 55-//3 7		·240			16125	-	YES	No	Do	De		
140	35.484 	2.000			-1	14	н	Yes	No	De	Da		
141	37-92/	2.000	¹ 240			-	HT .	YES	Yes	Two Distin	ICT EPLOSIONS		
192	64+165 42-164		. 2 4 0	-		-	- 11	Yes	Yes	D ₁	Da		
193		2.000	-840			15.625		Yes	No	ENTERNAL M.	ITERS EXPLANTE		
194	61-1-66-691	1 9-000		-		- 46		YES	Yes	Two DISTIN	CT EXPLOSIONS		
195	4110-46 94	0 2-000	-240					YES	No	EXTERNUL M	hervice Explosive		
.94		2-000	240	*				Yes	No	Do	D,		
197	37-121 28-62-55-64	2-210	*240		-16		L	Yes	Yes	3			
195		1 2-010	-240	1.0				Yes	Yes	4			
199		\$ 9.447		1 .		13.75		Yea	Yen	FLAME LE 44			
200	594.35- 379-1 7 67-17	1 2-600	-943	-	4	1.125	-09 6	Yes		ETTERNAL M			
dat		2-064	•24.0	+	-	1-50		Y _e s	No	De	Dr		
202	- 64-38 66-26	2 2 000	-240			3-0		Ye.	Ne	D,	P.		
103		240.00	+240	1	-	3-75		7. s	No	Do	Do		
214		2.440	+940		-	4.50		1	No		De		
205		2 D PUR	-#40	- u	1 -	5.00		Nes	No	De	D _c		
206	73-129-71-3	2 2 000	-976	<u>.</u>	•	5.50		1 Jan	No	O _p	D _e		
207	75 C.	5 2400		1.2	-	5 50		1	Ne	De	Do		
208		511-27-10	.350		-	-4			Yes				
209	H-11 \$3-10	5 2.000	- 3.50	1.		- +++		Yes	Yes				
210	1339-44-20	ii ii	030	3.4	· /	1.00	.051	Yes	Yes	The Drive	NET ROPARTS		
24	il c 16-11	2 2 700_	.030	- 3·0 - 3·9	1	1.25	1.004	1.00	IM-5	EXTERN	URE EXPLANTE		
11	· 55 · 81	<u>2</u> 2.700 3	11-0-50				+ -	V	Na	De	00		
113	915		. 130		-	1 -		101	199		Ð.		
214	94.30	23 2.700 2 . 7	. 536	and the second se		1.	1			De			
315	97.0	03 2·700 33	- 134			1.	+			Do	Do		
14	94.7	20 2.70	1 d t 5	-			T.T	4		1.1	D _p		
10	- 12	9 9 7 14 R 9 7 81			- 14	1	+ -			De Di			
	12.211	16 2.70	1.036		•	-	† -			L. Dall	Do		
21	5 74		- b.87	_	1.0	L.	1.5				De		
	0 10-6	44 97 1-10 0	0.51	3.5	-1	4		Yes	No	20	Ŭ _e		
25	1 104.2011.02	11 31 7 7	1030	3.5	1	-		1 _{ES}	No		2		
2	2 - 162	61		14-0	14	4		Yes	N		D.		
11	3 200	19/ 10-1-0-70		14-0	+			l Ve	NL	0e			
2	2 4441-544	<u> </u>	ភ	14-5		150							
2		4 <u>5</u> 1- 156									EXPENIE		
2	1 - 542	11 p 55 0.70		14+5	- 11	1.41				THE DIA	INCT EXPLOSIONS		
1	32.6	131 37 - 2 01	106. 10	14.5	-	-				EVTERNAL	MILTORE EARLINE		
	1 - <u>32.6</u> 	67	1 · 03	14.5		-	1						
1 L	아 콼	99/r	7	- 14-5	010	1 -		YES	Ye	s			

					-63	3-					
No	GAS MIRTHE GUE Cana Reading E.C.	IN Jack SB		Васарын он Сасара	G A	P	IGN 1-т ()	чол. Елес	Remar	KS.	
230	54-14-34756 AD-836 5-700	1032			15%	.056	No	Ne	Sher Cite	art in .	E.C.
231	10 56 6 43-366 (0-700	130					Yes	Na	Externa M		PLUSING
252	25-346 26-346 9-700	-030	35	1.			Yes	Yes			
233	79-013 1701				-10		Yes	N ₆	EXTERNAL N	Inrure E	XPLOSIVE
234		• b 30	3.5	÷.,		i.	Yes	No	D.	D.	
255	31-796 32-496 2700	064.	3.5	-14		-	YES	Yes			
236	56 526	-030	14.5	-	~	•	Yes	No	ENTERNAL M.	INTORE E	TPLOSIVE
287	1 37-255	- 8.32	14.5		-	1.4	Yes	No	D,	D _n	
236		-0.50	14.5	- 11	1375	-	Yes	Ne	De	D_e	
239		· 030	14.5	-			Yes	Ne	Do	D.	
30	65 103 2 700		14:5	100			Yes	No	ρ.	Da.	
-771	10.97 12.700		14:5		*	-	Yes	No	D ₂	Da	
142	7244-7244 71469 2.786 77469	- 07.6 - 	14.5					No	Do	D_a	
243	76-199 2-70		14.5	41		-	Yes		Do	Da	
244	75 26 2.72	7 •630	14-5		150		and the second second	No	Do	Do	
2.5	61-666 2-70 St-916	0.00	14.5		1	+		No	De	De	
-44	62 g/6 3000		14.5		1625	*		No	Do	D,	
-47	97-3671952 0	•030	14.5		-	-0-	Yes	No	Dn	0	
243	90.967 2.72	n - ado	1.X		+	÷	Yes	No	Da		'a
	92-867 2/701 18407-4867		3.2		<u> </u>		Yes	Yes	EITERNAL M		PLASTRE
- 17	1177 270	0 -012				-	Yes	No		D.	D.
15	State 1	0.	12.5	-		-16	Yes				
	11164 5.70	1010	+ 14.5	-	Ê		Yes Yes	No	Da-	De .	Do
25	3.864	e - 025	- 14-5	-	-		Yes	-No	Do Do	Do Do	.Do.
-5	c 6619	1030	- 14.5	-			Yes	No Yes	120	200	4-0.
25	9.000	101				- 10	Yes	No	ENTERNAL	MITTORE	EPRESIVE
15	1 12617-14012	.012	- 3.0	-		1.	_	No	Do	Do	
:22	COLUMN TO A REAL OF THE REAL O	1050	- 3.5			-	Ye.	No	Do	Da	
23	the second se	.125	3.5	•			Yes	Na	De	Do	
	0 - 7: 99 1 71 437 7 7	- 030	- 3.5		-			YES			
21	1	- 435	3.5	10.0				No	1	novie Es	PLOSIVE
	2 15 424 27	-633	3.5		-	-		Yes			
24	3 HAV-21 AM	-633 06	5.0	- 6	14	н		Yes			
	(A	00	3.5	-		- ÷	Yes	No	ELTERNAL M	INTORN E	PLADUC
	65 - 33 9/41 26 16 /4 27	·026	5.5	100	1.	-	Yes	Yes			
	6 - 36-646 59-366 =	+ 07	I 15.5		1.00	-	Yes	s Yes			
	17 Barrishing		430	1	1.	- 04 8	Yes	No	EXTERNAL M	INTURE E	XPLOSINE
	42245 2 1	1020	30		141	n	Yes.		<u></u>		20
	-1 AL 999 20	100	- 30			<u>р.</u>	-	Yes			Da
	<u>47</u> 735 2.7 71 47 749.	- 01	2 33	1-	10	-	Yes				Do
	1 JU . 69 2-7 79 JE 502	•o			-	+	Yes		10		De
	173 - 53 234	00 • 03				-	Yes	+ .			Do Do
	74 - 55.965	140 163					YEL	-			
	T5 55465 2.7	100C 7	195		-	-		No	- Pa		De
	61 307 2.	7/10	4 33		1	-	1155	N			00

-		_			_		- 47	•	_			4
W	GALI	MASER	Gas	14	BREADTH	Cuelent	G A	P	IGNIT	TIO N	REMARKS	
No	READ	I ML	EC	SB	FLANDE	Annices	INCHES	INCHES	HTERE	65514	nerianos,	
276	61. 3 47*	4.12%	2- 11	630	1.		1625	·058	<i>Jes</i>	Ye S		
277	- 4	k174 k33k	3 700	-024		55			YES	les	and the second se	
278		4 679 4 574	1-700	1025					YES)es		
174	1	294	1000	- 615	~	35			Yes	No	EXTERNAL MIXTURE EXPLOSIVE	2
250	1234640	125-171	8.504	1020		33			YES	No	D_0 D_0	
287	7	7.2.2.4		- 022		38			Yes	No	Do Do	
212	1	7-165		-023					Yes	No	Do Do	
243	1 5	0.597	2700	-024	1 - 1	33		+	Yes	No	Do Do	۲
284	1	3-4-4		-025		33		-	Yes	Yes		Η
	Rellin	12/16.2		1074		33			Yes	Yes		٦
235		2-81 ⁺	and the second se	174		35			Yes	No	EXTERNAL MIKTORE EXPLOSIVE	-
286	(V 514)-		\$ 700	-012					Yes	NO NO		-
387	34:312	46.536	2.160	.074		33				No	P. D.	-
268	99-41	67410410	2700	.034	-	29	÷ -		YES		0	4
259			2700	1824	-	49			YES	No	Do Do Do	4
440		3.936 3.96	2700	1224		49			Yes	No.	Do Do D.	4
<u> 291</u>			2.00			49		-	Yes	No	Do Do Do	4
292	6.042	P.LE	2.700			19	-		Yes	Yes		
.293		9:623	2 700	36	**	49			Yes	No	EXTERNAL MISTURE EXPLOSIVE	
194				035	- 11	19		-	Yes	No	De	
295				- 254		49			YES	NO	Do Do Do	
Ξ¥0		8. 13. 20.73	 2- 70 0	1.032	- 10	49			Yes	No	Da Da Da	
247		20-77/ Sandki			-	49	1.		Yes	No	Do Do Do	Π
298	1	23.505				19)es	No	Do Do	٦
149		<u>())</u>	- 19 76 o		1.	10)Es	Ye s		٦
1.1			9 700			4.0			YES		EXTERNAL MISTURE EXPLOSIVE	٦
30	1 2	7705				40		**	Yes	No	D_0 D_0 D_0	٦
305	7			0.32	- 1	AD			Yes	Ne	no Do Po	٦
30	and the second second	D 527	6		4 1.	40			Yes	No	D_0 D_0 D_0	٦
59		30.04) 	12		+ -	40	t F	1.	Yes	Yes		H
30	1 13 09			· dE	4	40			Yes	No	ERTERNAL MINTURE EXPLOSIVE	۲
	6 43.8	0458	3 2-70	12155	2	40	+ -		YES			+
30			2 2-70 0	- 132	5	40		11	YES	No Yes		4
50			5 74	-032			- -				EXTERMAL MISTURE EXPLOSINE	4
50	4 56 -2		2 700	· 034		26	+	-	Yes Yes	No	Do Do	4
54	0		2 270 2	03,	3	50		-		No Yes		
31			े २ २०। इ	· 03	_	50	1 -	-	Y _{EJ}	No	ENTERNAL MINTURE EXPLOSIVE	H
\overline{h}			8 4-70	· 03		1	+ -		Yes	No	Do Do	-
			77 2.78 33	· 03.	3	50			-	-		-
			3 2-7 1		2	50			Yes Yes	No V	Do Do	-
3			270		- 11	23		4	Yes V		EXTERNAL MILTURE EXPLOINE.	-
	16	73-19	u - 276 301		2	23	+ -	**	YES	No	The second se	+
	19			- 4 3	-	23	+ -	1-	Yes	No		
	18 75-6	25.66	2 2 Z	0 -01		23	+ -	-	Yes	No		-
	19	-	19	• • •		23	+ -	+	Yes	A/c	A	
	20					23	+ -	-	Yes			_
	27 H.J		2.78	·03		16			Yes			
		44	2 2076	-01	4 -	16			Yes	Yes		

	-65-													
No	Gas Mena Russing	Gas E.C.	;"" 56	C.ICAT IN Aminis	Becom of Games	G A	P	lenia Innia		Remar	KS			
322	2.558		·#3 2	16	1-	1-625	.058	Year	No.	EXTERNAL M	INTORE EXPLOSIVE			
325	0.5.6P.9	2.5	-057	/6	4	- 42		YES	Y≓s					
384	95 <i>071-9518</i> 2 47-103	7.708	-032	16		*		Yes	No	ELTERNAL I	FITTORE EXPLOSIVE			
325	101- <u>12-9</u> 6 101- 1 21		90.4	60	4	-	-	Yes	Ne	Do	Do			
3.26		25700	-034	60	~	-	-4	Yes	No	Do	De			
377	7.101		-131	60	-14	-		Yes	Ne	0.	D+			
328	9.231		-34	60	9	**	1÷	Yes	No_	Du	De			
324				60	1			Ye s	No	D.	De			
330	15-217		10.21	60					No	Do	Do			
53/	17.965	In the second	-0:24	60		-	-	Yes	No	De	<u>De</u>			
	70665		025		÷ "	÷	-	Yes	No	Do	Do			
	23.396	2.700	125	60			-	Yes	No	Do	De			
155	96-155 21-14		-044	STATISTICS.		1		Yes	Yes	Formar M.	TURE EIRESIVE			
				78	1.	-		Yes	No	Do	Do			
357	- Inhear			78		-		Yes	No No	De	De			
335	<u> </u>	2.760	-055	78		-		Ves	No	Do	Do			
1559	31.094	2.700			1 6	<u> </u>		V.	NE	Do	Δa			
	39-128		- 113,					Yes	No	Do	De			
	23.52	2.780						164	10					
				GONL	Guner	64		IGNIT	now	Rena	085			
				Genits	141	на та Імсия в	Derth	interes	Filme	REMA	KN3			
	4550	d.	- 172		20	1.6.25	-061	Yes	710-		D Between Frances			
543	a - 1 - 5 - 4	5 2.700				- 4	.059	Vez	No	D. D.	and the second states and			
					1			Yes	No	Do Do	$P_{a} = D_{a}$			
	- 5¥2.9	Nic.			15		051	Yes		Bear Barnets file Fales tomater No.	ant Banne Der Benetan all Franz Smith Lean			
		6	-621						N/4	0,	De			
34		e No c	- 223	4	1	1 -		Yes	Nit	Do For Artonna P	PERE PASSED			
34			03:		41			Yes	N.4		De Or			
34		Nic.	-074	يدار والكارات	-	1.64	-	Y_{t+1}	Net	Burtheavy the	Do Do reflect in Ernation			
<u>54</u>		W4L	-626	<u> </u>	-			Yes	Nº 4					
		100	87			1		Ye:	N/L	LESS YIDIEN	T EXPLOSION			
23	-	Alte	02	-	31	+		Yes	Na	D,	0 ₀			
1		NIL	-Br	1			-	Yes	Nin	Dett & IMALE				
	the second se	Non		- • !		е		Yes	Ma	REMAINING	GUANTITY OF OULT			
	15 12 523- 57.5	1005	- 014		-	-	-	Tes.	Non	EXTERNAL M	ITTURE EXPLOSIVE			
II 6		1970	102	- 3.5 m		-	14	Yes	Ne	FREGLE INT.	ERMAL PELISION			
	57 MARINE (24)	2 2.74	- 02	1.1		+-	+	Yes	No	ENTERNAL M	ATTARE EXPLOSIVE			
		1 2.70	-02		-			Yes V	-		NGCS CONSENSED			
3	30 430	2 - 7n	0 .07	1			÷.	Yes Yes	Yes.	EXTERNAL MI				
	60 20.73	\$ 74	10.2	-	+	-	-	Yes	1		De De			
	1	14.74	102	and the second second	12	+	01.8	Ver Ver	Ne	CALER OF B	BOS BRUKEN BY FORCE			
	1- IT: 7-77 A		1.02		-	-	054	Yes	No	OF EXPL	AJION AINTURE ENPLOSIVE			
	6 : 12 4.7 802		° .02		· ·	1.		Yes	No	\mathcal{D}_{μ}	0+			
3	164 53.1	24	112	-	1	-		Yes	Na	0,	D _e			
	120	alle ni	-		1		-	1.1.1	1.14					

						20-						
	c	GAS	14	CIAL	CLINERAT	GA	P	IGNIT	', d N	REMI	2110	
No	Gas Mover	EADA	feet	Dust			Perra	in the stand	Elfrent	NEMI	AKKS.	
	READINE	EC	5B	G. RAM S			INCHE_	v		(CARDING)	N	C. Martine
365	88.617 Reis-1164		124	10.5 — [15	1625	-058	Yes	N.	ERTERNAL	"Jur uRE	and the second second
366		2700	And in case of		÷1			Yes	No			D.,
567	- 068	2700	-075			5		Yes	No	Da		
365	91.126	2700						YES	No	P.		Do
369	94-551 94-551	2714	-076	3.5 maro			-	Yes	No	Do	Da	Do
370	99-552-99-577 142-777	2.700	1025	H	+		-410	Yes	No	0.	D.	
37/	102.30		1037					Yes	No	A	De	D.
372	120229,2-14	A DESCRIPTION OF	1626		•	•		Ye.	No	Do	4	Do
373	19.864		.054	-		-	+	Yer	No	D.	Da	De
374	22.570		1076	+		4		Yes	No	Do	D _o	Da
375	44.397 28.814.28.114		.027	- 4.9	**	-		Yes	Na	0,	De	Do
576	11-130 20-020		-027,	3.5		-		Yes	No	D.	Da	Ďo
377	31.90		076			- 0.	++	Yes	No	D,	Do	Do
78	51449-13994			1.11		1.		Yes	Yes	Cover Or Box		By Forte OF
	3812 67 31-77		020					YES	<i>⊺∈s</i> No	ELTERNAL M		PLOSIVE
		71	102.9	- ADMO	10			Yes	No	b.	D.	0
		1-700	1.079		1.			Ves	NO NO	-	P6	
										D ₀ Do	De	
385	47-461 47-82 - 47-57				-			Y _{EJ}	Na	0,	Р С 0.	D
080	the second se					**	e .	YES	No	GATA THATE I		OF DUST LEFT
		5.400	-030	NONE		1.	11-	Yes	YES			
313			.078	- +1	•		*	Yes	Yes			
386			1.070	+	"	**	.41	Yes	YES			
76	- 61.61 j			.4.) H				Yes		M	6
			1078	3.5	+-		H	YES	No	EXTERNAL		CXPLOSIVE
				1 *		1-		Yes	No	Dø	Do	Do
			-629) <u>e</u> 1	NU	Do	Da	
<u>39</u>		6 0 2.000	9	- 4		- 441	6	$\gamma_{\epsilon s}$	No	Q_{0}	De	Do
<u>59</u>	2 72-73	0' 0 9-700	030 U		20	- 11	<i>i</i> 4 -	Yes	No	$b > b_{i}$	Do	
39	75.46	9 8 19 740	·033	 		+0	+	YES	No	120	De	
39	4 X \$*/91	4 2·70	0 · 031	<u> </u>	•	-4	•	YES	No	Do	Do	Do
34	5 20 900- 809	21 2.701	.03/	+ +	••	ų.	-	Y⊨s	No	Po	Do	Do
34	4 <u>63·66</u>	60 2 70	_!•o2g	++	- 12	••	.,.	Yes	YES			
34	H 252 (-163)	912-20	-055 0	- 3-480		10	-	Yes	No	FXTERNAL	MIXTUR	E ENPLOSIVE
34	S 1451/10 19-		030		+		10	Yes	No	Do	D٥	Do
3í	14 41 S 1		031		-	-46	-	Yes	No	Do	Do	Do
<u>k</u> (10 96.550-96+	10 	·03c	4 +		44		Yes	NIL	RED SPARES	AND FLAT	TES AT GAP
k	§1 92.61	- Mik	1.03	4 .	-			YES	NIL	D۵	Du	,
	02 91.6	72	· 03	1	-	-		$\gamma_{e,j}$	NIL	Do	D	,
1 Li	03 1,10-100 0		. 03:	- 3 5 - 5	-	1.	1.	Yes	N.L	Less F		
	04 100-0		+834	- ANE				IES.	1		0	
	105 100 00	NIA	-030						Nit			
1	06 100-13	2 10	-03	3.5 ANO					Nr5	FAIGLY NIL	Do LENT. 5	NART WHITE
	107 100-180 100	407 195		5 400	25					FLAME AT	OPENIA	6
	09 100 23	3	• 045							VIELENT		
	609 100-20	ALC:	104							LESS Vie Two REPO	RTA HEAD	PLOSION RD.
	410 100-30	N-L	4									
		N	E	" <u></u>	1					De	Dø.	

	-67-													
GAS METER CUBIC Feet DUST IN READING EC S8							G A	DEPTH	IGNI Inteenal		Remarks.			
14	1402.4	- 14.5		.042	5.5 <u>-</u>	25	1.625	· 0.52	Y	NIL	RED FLAMES AT GAP.			
112		10	au_	106.0					YES	Nu	VERY WEAK EXPLOSION.			
1.5		15			-				Yes	Niz				
414		ela.		- 14			-			N.E	VILLENT EXPLOSION			
115	J40.8	T FOR SU	Nie Nie	10.57		•			Yes	Net	2-04			
11	ler ĝij	160.94	N11 -	- 852	-11-		1n	-	Ye:	AL	the second se			
117	144.952	-/15-941	N-L	.030-	5.5		-	4.	Yes	Net	WILLENT EXPERING N THE AMENNT OF FRAME AT GAP			
4.18	1	117	N/+	-037				4.0	Ves	Nic	Do Do			
119		-061	Net	·044	-11		- 40		Yes	Nil	FLAME SEEDS SHAPPY			
120	F	-141	NUL	44.0		10		-	¥€.	Nil	Do Do			
121		126	Non	126	10	$ \cdot $			γ_{e1}	Neck	FARAF VIALENT FAIR AMOUNT			
1.55		- 16	Nik	-011				-	Yes	Net	Da Da			
-13		-11	Non	-22.5	14	-	-		Yer	NIL	Do Do			
424		124.50	N/F	1425			-910	0	Y	Net	Do Do			
-125		1.175	Nim	-025	- Jugo	1+				Nik	Do Do Same Duge Mature Experience			
476			2.700	125	- 17			- 11	Yes	No	EXTERNOL MET			
-17	1.00		2700	1.010			-		Yes	-	ELTERNOL MIRTURE EXAMINE			
-24	-	6-740 9-4 80	2.74	- 030			- 16	-	YEI	No				
17:24		12 2 10	2.74		5 Junio	· ·	-	-	YEI	No	Do Do Do			
130	4	1-017	1.76	1-030	. 10	1 "			Yes	Yrr_	LINE SE FLED, I GALTING EST MANUEL MARK IN SA MATACH BUT AND SE SAND NARA BOX OFFICE, UNIX OFFICE L BANG BOX OFFICE, MARK ENTRALINE			
-151		1-1501 17-71 20-115	2.760	1030			+	1.0	Yes	No				
43	•	2045 1045	1 2 700			17	*		Yes		D_0 D_0 D_0			
43	-	13-16	2.74	u Julio		20			_	Na	Da Da Da			
4.5	_	24 00	3 079 11	0 1030		30		**	Yes		Do Do Do			
-15		26 73	1 2 70	a	-	40	- 10	1.0	Yes		Do Do De			
45	6	27 4 15-3110	3 270	0 1-035		60	<u>, "</u>		Yes		De Do D- Dust ALL CONSUMED			
13		15 AF 11 - AF - 0	0 270	10	-	60		1	Yes		FEEBLE INTERNA, ENROLION			
7.2	-	34-11 60-11-9	55 2 70	-64.3	- "Add	(n) - 0.	÷	-	Yes		EXTERNAL MINTURE EXPLICITIE Do Do Do			
.43	7	1011	1.7 12 20	0			+ **	÷	Yes Yes	No	HULLENT INTERMAL ENTERING			
L	1 67	42.34 14 - 44	43 2 70	0 +040	1	1		-	YEI	No No	LEIS VILLENT IN BRIAN EIDLESN			
L	-	198-25-	16 270 01.	0.13		1 1		1.	Yes		VIALENT INTERNAL EXPERIMENT			
-4	_	- 11	12 = 10 17 1 - 272	-07	4	1 11			Ye	-	Do Do Do			
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54	60-696	Caller of	als						Yes		AT URE	EXALINE	
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559	5-11 VA SH		1050		L.C.	-	4	YES	N _D	010 803 W			e Do
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593		2700	.030	ę.	-44	•	+	Yes	No	Ď۵	D.	Do
594	42-410		-020		**		•	Yes	No	٥,	Do	D.
595	45.640 91-340			*		4		Yes	No	Ber Bitmin	Bex Com	
596	1 4 Stor - 9 3d 10 230		1991		4.	4375	••	Yes	No	Do	Do	Do
597	3-960	2.700				1.500	•	Yes	Yes	Two Dis	TINGT Ex	PLOSIONS
598	3155	2.700	i d Bg		+	•	-	Yes	Yes			
599	6400-6-830 R-350	2760	•03 <i>0</i>			•	.050	Yes	No	EXTERNAL [OLD		EXPLOSIVE ND CONERT
	12.250	2.700	• <u>030</u>	• •	~		•	Yes	No	Do		Do
601	19.310 15.010	2.760				•	•	Yes	No	De		Dø '
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603	10	ant de					.052		Yes			
604	23.700				•	PERMIT			Yes	IN FLANGE	S To Give 1	
605	23.730		•430						Yes	AND SIDES	OF AND OF	PED ATENDS SREIPECTIOFLY
606	20141-16-710	241		++	1.4	- +	- 14	Yes	No	EXTERNAL FILED DOWN	MITTURE E: To REDUCE	GAP
60	32.140							Yes	No	0	0	Do
608	32.170		-030	<u> </u>			*	Ye+	No	D	0	Do

E.C. MEANS EXPLOSION CHAMBER. S.B. MEANS SWITCH BOX.

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INTRODUCTION:

In 1927, the Committee of The Journal of the Royal Technical College, accepted for publication an account of part of the work. A copy of the reprint is herewith presented, together with a mathematical investigation given by way of extension to this paper. The author acknowledges his indebtedness to Professor James Muir, M.A., D.Sc., A.R.T.C., and Mr. Alex. D. Third, B.Sc., Ph.D., A.R.T.C., for assistance given in the mathematical investigation. The drop in Temperature produced by the passage of Hot Gases through apertures.

The following rough calculations are given by way of extension to the paragraph at the foot of page 119 of the 4th. Number of the Journal of the Royal Technical College (Faper on Preventing Electrical Apparatus from starting Explosions).

The paragraph is as follows:- "The cooling of hot gases, by passing through a gap, may be regarded as being effected in two ways (i) naturally by the sudden expansion from high pressure to low, and (ii) by passing over the cold surfaces of the metal flanges. The heat absorbed by a cold metal surface from a hot gas passing over it, is proportional to the area of the surface, to the density of the gas, to the speed with which the gas sweeps over the surface and to the difference in temperature between the surface and the hot gas. When a 'gap' is short, deep and of small area the first cause of cooling probably predominates; with long narrow gaps cooling is possibly accomplished mainly by the large metal surface swept over."

In considering the passage of hot gases (such as steam) through nozzles, it is found that the passage of the gas may be regarded as being a "reversible adiabatic expansion". This may be a large assumption if applied to the passage of flames from an explosion but it will be made (combustion is not adiabatic). For the reversible adiabatic expansion of ideal gas, $pv^n = \text{Constant}$ and pv = RT, hence if the expansion through the gap be from P_1 , T_1 , to P_2 , T_2 , $T_2 = T_1 \left(\frac{P_2}{P_1}\right)^{\frac{n}{n}}$. Taking n (the ratio of the specific heats of the gas at constant pressure and constant volume) as 1.4 and $r = \frac{P_2}{P_1}$, then $T_2 = T_1 r^{0.286}$. If the gap is a convergent or a parallel one the outlet pressure is always 0.53 of the initial pressure (if the final pressure is not above this). If the passage is divergent, fuel expansion can take place within its boundary.

Should/

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Should the velocity of the stream at any point in the gap be required it is given by the equation,

$$= \sqrt{\frac{2gn}{n-1}} P_1 V_1 (1 - r\frac{n-1}{n})$$

where P₁ is in 1bs/sq. ft.

 V_1 the specific volume in cub.ft/Lb.

r the pressure ratio at the point (referred to P_{τ})

g the acceleration due to gravity.

or the flow is given by

 $w = 2 \sqrt{\frac{2gn}{n-1} \frac{P_1}{V_1} (r^{\frac{2}{n}} - r^{1+\frac{1}{n}})}$ where w is the flow in 1b. per sec. and 'a' is the area to flow in sq. ft.

CASE I. The extreme case of the gap $\frac{1}{10}^{"}$ x $\frac{$

The form of the passage is of the same order as in nozzle work, the heat transference through the limited surface of metal is negligible when the quantity of gas flowing is considered; the expansion being regarded as adiabatic we have |

 $T_2 = T_1 = 0.286$ $T = \frac{P_2}{P_1} = 0.53$ so $T_2 = 0.83T_1$

If say $T_1 = 2000^{\circ}F$ (absolute)

 $T_2 = 1660^{\circ}F \text{ a drop of } 340^{\circ}F$ but for this case Fig. 5 p 122 of the Journal paper gives a maximum Pressure $P_1 = 57$ lb/in.² (abs). So $P_2 = 0.53 \times 57 = 30$ lbs/in² $P_3 = 15$ lbs/in² (atmospheric.)



There is thus expansion from 30 lbs to 15 lbs/in² after leaving the gap. This expansion takes place very close to the gap and produces further cooling which is probably of importance. CASE II. Another extreme case - the gap 0.03" x 29" x 1"



Now consider the heat lost by transmission to the cold metal sides of the gap.

Reynolds' laws of heat transmission, stated in the paragraph quoted at the beginning of this note may be expressed by the equation.

H = B / vtm [H = A + B / vtm - the term A for gases at rest being negligible.] where tm is the mean temperature difference between hot gas and metal; v is the speed of gas, / the density, B a constant. H is the heat transmitted per unit area per second.

If # = total weight of gas passing per second and

a = cross-section area of flow, $w = a v /^{\circ}$ and

 $H = \frac{w}{a} Btm$ (Heat transmitted per sq. foot per sec.) For ordinary transmission through boiler tubes $B = \frac{1}{1200}$ to give B in B.T.U. per sq. ft. per sec., w being in lbs/sec, a in sq. ft., tm in ^OF.

Now T_1 and T_2 being the temperatures of gas on entering and on leaving respectively, then (ignoring natural drop due to expansion) we have

Heat taken from gas = w.S. $(T_1 - T_2)$ where S is the spec. heat of the gas (pressure constant).

So w.S. $(T_1 - T_2) = \frac{\pi}{a} B A tm$	where A _ surface area of metal which is
$T_1 - T_2 = \frac{B.A.tm}{B.a.}$	independent of flow of gas as experiments

Substituting numbers and taking 8 = 0.24 and the temperature of the hot gases somewhere over $1800^{\circ}F$ (absol.) and temperature of metal over $500^{\circ}F$ (abs.) so that tm = $1300^{\circ}F$, we get (A and a, both being put in sq. inches.)

$$T_1 - T_2 = \frac{29 \times 2 \times 1300}{1200 \times .24 \times .03 \times 29} = 300^{\circ} F$$

The total drop in temperature might thus be about $500^{\circ}F_{r}$ - 200° due to natural cooling (expansion) and 300° due to transmission to the metal.

Of course the various assumptions made render the calculations of very doubtful value. The <u>experiments</u> might lead one to expect that the drop of $340^{\circ}F$ got in Case I should be more nearly equal to the $500^{\circ}F$ got in Case II (both gaps were flameproof), but the further cooling in Case I just after the gas leaves the gap could reasonably be taken to account for this difference, without having regard to a possible difference in the internal temperatures (in one case pressure attained $\frac{51}{N}$ lbs., in the other 21 lbs/in² abs.) But 500° seems too small a drop in temperature. If so, then it can only be suggested that calculations based on adiabatic expansion of ideal gases do not apply to gases in a state of combustion.

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APPENDIX 11.

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